

Article

Free Vibration of a Taut Cable with Two Discrete Inertial Mass Dampers

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Abstract: Recently, inertial mass dampers (IMDs) have shown superior control performance over traditional viscous dampers (VDs) in vibration control of stay cables. However, a single IMD may be incapable of providing sufficient supplemental modal damping to a super-long cable, especially for the multimode cable vibration mitigation. Inspired by the potential advantages of attaching two discrete VDs at different locations of the cable, arranging two external discrete IMDs, either at the opposite ends or the same end of the cable is proposed to further improve vibration mitigation performance of the cable in this study. Complex modal analysis based on the taut-string model was employed and extended to allow for the existence of two external discrete IMDs, resulting in a transcendental equation for complex wavenumbers. Both asymptotic and numerical solutions for the case of two opposite IMDs or the case of two IMDs at the same end of the cable were obtained. Subsequently, the applicability of asymptotic solutions was then evaluated. Finally, parametric studies were performed to investigate the effects of damper positions and damper properties on the control performance of a cable with two discrete IMDs. Results showed that two opposite IMDs can generally provide superior control performance to the cable over a single IMD or two IMDs at the same end. It was also observed that attaching two IMDs at the same end of the cable had the potential to achieve significant damping improvement when the inertial mass of the IMDs is appropriate, which seems to be more promising than two opposite IMDs for practical application.

Keywords: stay cable; vibration control; hybrid control; inertial mass damper; viscous damper

1. Introduction

With the flourishing development of materials and construction technologies, civil engineering structures are becoming larger, lighter, and more flexible, especially for long-span bridges. Cable-stayed is a common option for bridges in the medium to long-span ranges due to its unique structural formation, economic advantage, and esthetic value [1]. However, as important load-bearing components of cable-stayed bridges, stay cables are highly susceptible to dynamic excitations due to their high flexibility and low intrinsic damping [2,3]. Frequent and excessive amplitude cable vibrations may lead to fatigue failure of cables. These problems may inevitably shorten the service life and cause the risk of losing public confidence in cable-stayed bridges. To guarantee structural safety, several solutions have been proposed to dampen cable vibrations, which include modifying aerodynamic surface of cables [4], connecting cables together via cross tie [5], and attaching external dampers on cables [6–9].

Though these practical measures have been well applied in the field, each has its own shortcomings. Changing the surface of the cable is difficult to implement for retrofit and may increase drag forces at

high wind velocities [10]. Cross-ties are incapable of direct energy dissipation and make the aesthetics of cable-stayed bridges deteriorate [11]. Compared to the two methods above, attaching external dampers on the cable seems to be more promising. Nevertheless, the installation location of a passive viscous damper is typically restricted to within a few percentage points of the cable length from the cable anchorage [12]. As expected, passive viscous dampers cannot provide sufficient damping to eliminate vibrations for super-long cables, such as the Sutong Bridge, with cables nearly 600 m long. Moreover, the results based on both theoretical and experimental studies indicated that the existence of the cable sag [13,14], the cable flexural rigidity [15,16], the damper stiffness [17], and the damper support stiffness [18] or their coexistence [19–23] would have adverse impacts on the efficiency of passive viscous dampers.

An active damper can produce a force-deformation relationship with the negative-stiffness behavior that benefits damper efficiency when the linear quadratic regulator (LQR) algorithm is employed [24,25]. However, active dampers often require high power sources beyond practical limits and are thus rarely used for cable vibration mitigation in real bridges. Alternatively, semiactive dampers, which can produce similar hysteresis and achieve control performance comparable to that of active dampers, were proposed [26–29]. For instance, the semiactive control based on magnetorheological dampers has been successfully applied on the Dongting Lake Bridge [30], Binzhou Bridge [31], and Sutong Bridge [32]. Compared to active dampers, semiactive dampers require less power. Nevertheless, possible implementations of semiactive dampers on site still require an external stable power supply, a sensing system, and a controller, which seems to be complicated and costly. This fact has inspired researchers to introduce a negative stiffness mechanism into passive dampers to mitigate cable vibrations.

Recently, several representative passive dampers with negative stiffness mechanisms, including pre-spring negative stiffness dampers (pre-spring NSDs) [33,34] and magnetic negative stiffness dampers (magnetic NSDs) [35,36], have been successfully developed. Negative stiffness dampers have well demonstrated to be capable of providing superior damping over that of traditional passive viscous dampers [37–39]. However, extremely large passive negative stiffness may make the NSD lose its stability. Alternatively, an inerter has the potential to provide similar negative stiffness without a stability problem [40]. Many inerter-based absorber layouts have been proposed, and their control performance advantages have been proven for civil engineering structures [41–59]. As for the vibration suppression of cables, typical inertial mass dampers (IMDs) [60–65] and tuned inerter dampers [66,67] were well developed, and their significant improvement on the achievable modal damping ratio of the cable was verified via both theoretical and experimental investigations.

With the increasing cable length, it may be difficult to attain a desired level of supplemental modal damping with a single damper or a pair of dampers installed near the deck anchorage. Hence, some hybrid techniques have been further proposed. The idea of combining external dampers with cross-ties for cable vibration control was considered, which not only addresses the deficiencies of these two commonly used countermeasures but also still retains their respective merits [68–73]. A hybrid damper system, combining a viscous damper and a tuned mass damper, can overcome the shortcomings of single type of dampers and improve effectiveness and robustness in suppressing cable vibration [74]. In addition, application of two viscous dampers or two high-damping rubber dampers at different locations of a cable was proposed [75–77]. The results have shown that when two viscous dampers are installed at opposite ends of a cable, their damping effects are approximately the sum of the contributions from each damper [77]. However, when they are at the same cable end, the maximum modal damping ratio of the cable is determined by a single damper at the further distance, indicating no benefits over a single damper configuration [77].

Inspired by the potential advantages of attaching two external discrete viscous dampers (VDs) on a cable, this study aimed to evaluate the feasibility of a cable with two discrete IMDs, either on the opposite end or on the same end of the cable, to improve the vibration mitigation performance of the cable in each mode. Complex modal analysis based on the taut-string model was employed and

extended to allow for the existence of two external discrete IMDs. The formulation for free vibration of a taut cable with two discrete IMDs was established, and corresponding complex wavenumber equations of free damped vibration were derived. The asymptotic and numerical solutions of the wavenumber equation were obtained, and the applicability of asymptotic solutions was then evaluated. Finally, parametric studies were performed to investigate the effects of damper positions and damper properties on the control performance of the cable with two discrete IMDs.

2. Formulation of the Cable–IMD System

A taut cable with two transversely attached inertial mass dampers is shown in Figure 1. The length, the mass per unit length, and the tension of the cable are L , m , and T , respectively. The coordinate system defines that the x -axis and the v -axis are along the cable chord and the transverse direction, respectively. $x^* = L - x$ represents the coordinate from the right end of the cable. Two discrete IMDs are respectively installed at distances x_1 and x_2 from the left end of the cable ($x_2 \geq x_1$). The distance between the right IMD and the right end is denoted as $x_2^* = L - x_2$. The damping coefficient and inertial mass of the j^{th} IMD are denoted as c_j and b_j ($j = 1, 2$), respectively. The equation of motion of the cable–IMD system is given by:

$$T \frac{\partial^2 v}{\partial x^2} - m \frac{\partial^2 v}{\partial t^2} = \sum_{j=1}^2 F_{IMDj}(t) \delta(x - x_j), \tag{1}$$

where $v(x, t)$ is the cable transverse displacement and $\delta(\cdot)$ is delta function to specify the location of the damping force F_{IMDj} at $x = x_j$.

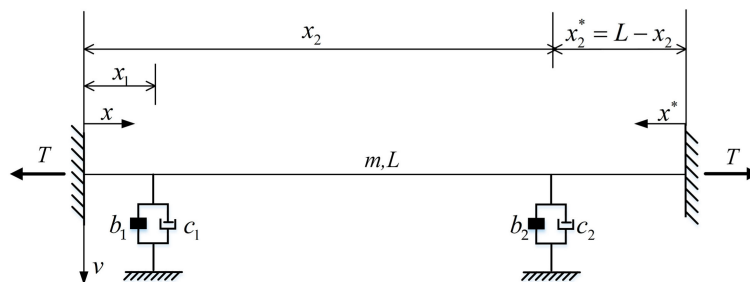


Figure 1. The taut cable with two discrete inertial mass dampers.

Under free vibration, applying separation of variables, the cable transverse displacement and the IMD force can be respectively expressed as:

$$v(x, t) = \tilde{v}(x)e^{i\omega t}, F_{IMDj}(t) = \tilde{F}_{IMDj}e^{i\omega t} \tag{2}$$

where $i^2 = -1$, ω is a complex natural frequency of the cable, and $\tilde{v}(x)$ is the corresponding complex mode shape. To find $\tilde{v}(x)$, the cable can be considered as a multispan structure connected at the damper locations [77]. Substituting Equation (2) into Equation (1), $\tilde{v}(x)$ in each span needs to satisfy a homogeneous equation [77]:

$$\frac{d^2 \tilde{v}}{dx^2} + \beta^2 \tilde{v} = 0 \begin{cases} 0 \leq x \leq x_1 \\ x_1 \leq x \leq x_2 \\ 0 \leq x^* \leq x_2^* \end{cases}, \tag{3}$$

where $\beta = \omega \sqrt{m/T}$ refers to the wavenumber.

Applying boundary conditions at cable ends, i.e., $\tilde{v}(0) = \tilde{v}(L) = 0$, and the transverse displacement compatibility conditions at damper locations, i.e., $\tilde{v}(x_1) = \tilde{v}_1$, $\tilde{v}(x_2^*) = \tilde{v}_2$, the general solution of Equation (3) can be further written in the form [77]:

$$\tilde{v}(x) = \begin{cases} \tilde{v}_1 \frac{\sin \beta x}{\sin \beta x_1} & 0 \leq x \leq x_1 \\ \tilde{v}_1 \frac{\sin \beta(x_2-x)}{\sin \beta(x_2-x_1)} + \tilde{v}_2 \frac{\sin \beta(x-x_1)}{\sin \beta(x_2-x_1)} & x_1 \leq x \leq x_2, \\ \tilde{v}_2 \frac{\sin \beta x^*}{\sin \beta x_2^*} & 0 \leq x^* \leq x_2^* \end{cases} \quad (4)$$

where \tilde{v}_j is the mode shape amplitude at the j^{th} damper location.

At damper locations, there is:

$$T \left(\frac{dv}{dx} \Big|_{x_j^+} - \frac{dv}{dx} \Big|_{x_j^-} \right) = \tilde{F}_{IMDj}(t). \quad (5)$$

Substituting Equation (4) into Equation (5), it yields:

$$\begin{cases} \cot \beta x_1 + \cot(x_2 - x_1) - \frac{\tilde{v}_2}{\tilde{v}_1} \frac{1}{\sin \beta(x_2-x_1)} = -\frac{\tilde{F}_{IMD1}/T}{\beta \tilde{v}_1} \\ -\frac{\tilde{v}_2}{\tilde{v}_1} \frac{1}{\sin \beta(x_2-x_1)} + \cot \beta(x_2 - x_1) + \cot \beta x_2^* = -\frac{\tilde{F}_{IMD2}/T}{\beta \tilde{v}_1} \end{cases} \quad (6)$$

Substituting \tilde{v}_1/\tilde{v}_2 from the second one into the first one of Equation (6) and rearranging, the characteristic equation of the wavenumber β is derived as:

$$\begin{aligned} & (\cot \beta x_1 + \frac{\tilde{F}_{IMD1}/T}{\beta \tilde{v}_1})(\cot \beta x_2^* + \frac{\tilde{F}_{IMD2}/T}{\beta \tilde{v}_2}) + \\ & (\cot \beta x_1 + \frac{\tilde{F}_{IMD1}/T}{\beta \tilde{v}_1} + \cot \beta x_2^* + \frac{\tilde{F}_{IMD2}/T}{\beta \tilde{v}_2}) \cot \beta(x_2 - x_1) = 1 \end{aligned} \quad (7)$$

3. Two Opposite IMDs

3.1. The Wavenumber Equation

The damper force of the j^{th} IMD can be expressed as [45]:

$$F_{IMDj}(t) = b_j \frac{\partial^2 v(x_j, t)}{\partial t^2} + c_j \frac{\partial v(x_j, t)}{\partial t} \text{ or } \tilde{F}_{IMDj} = -b_j \omega^2 \tilde{v}_j + c_j \omega \tilde{v}_j. \quad (8)$$

When two IMDs are installed at different ends of the cable, substituting Equation (8) into Equation (7) and using trigonometric relations, Equation (7) can be rearranged to the form relating to x_1 and x_2^* as:

$$\tan \beta L = \frac{A_1 + iB_1}{C_1 + iD_1}, \quad (9)$$

$$A_1 = -\chi_1 \sin^2 \beta x_1 - \chi_2 \sin^2 \beta x_2^* + (\chi_1 \chi_2 - \eta_1 \eta_2) \sin \beta x_1 \sin \beta x_2^* \sin \beta(x_1 + x_2^*), \quad (9a)$$

$$B_1 = \eta_1 \sin^2 \beta x_1 + \eta_2 \sin^2 \beta x_2^* - (\chi_1 \eta_2 + \chi_2 \eta_1) \sin \beta x_1 \sin \beta x_2^* \sin \beta(x_1 + x_2^*), \quad (9b)$$

$$C_1 = 1 - \chi_1 \sin \beta x_1 \cos \beta x_1 - \chi_2 \sin \beta x_2^* \cos \beta x_2^* + (\chi_1 \chi_2 - \eta_1 \eta_2) \sin \beta x_1 \sin \beta x_2^* \cos \beta(x_1 + x_2^*) \quad (9c)$$

$$D_1 = \eta_1 \sin \beta x_1 \cos \beta x_1 + \eta_2 \sin \beta x_2^* \cos \beta x_2^* - (\chi_1 \eta_2 + \chi_2 \eta_1) \sin \beta x_1 \sin \beta x_2^* \cos \beta(x_1 + x_2^*) \quad (9d)$$

where $\eta_j = \frac{c_j}{\sqrt{mT}}$ and $\chi_j = \frac{b_j \omega}{\sqrt{mT}}$ represent the dimensionless damping coefficient and the dimensionless inertial mass of the j^{th} IMD, respectively.

The form of Equation (9) is suitable for solutions, either for asymptotic solutions or numerical solutions by iteration.

3.2. Asymptotic Solution

The following assumptions are introduced [12,17,77]: (1) The locations of IMDs are very close to the ends, i.e., $x_1, x_2^* \ll L$; (2) the wavenumber β_n of each mode n ($n = 1, 2, \dots$) has a small perturbations $\Delta\beta_n = \beta_n - \beta_n^0$ from the undamped value $\beta_n^0 = n\pi/L$. The assumptions above lead to the following approximations:

$$\tan(\beta_n L) \cong \beta_n L - n\pi \sin(\beta_n x_1) \cong \beta_n^0 x_1 \sin(\beta_n x_2^*) \cong \beta_n^0 x_2^* \cos(\beta_n x_1) \cong \cos(\beta_n x_2^*) = 1. \tag{10}$$

The asymptotic formula for the wavenumber β_n takes the form:

$$\beta_n L \cong n\pi + \beta_n^0 \frac{E_1 + iF_1}{G_1 + iH_1}, \tag{11}$$

$$E_1 = -\bar{b}_1 x_1 - \bar{b}_2 x_2^* + (\bar{b}_1 \bar{b}_2 - \bar{c}_1 \bar{c}_2)(x_1 + x_2^*), \tag{11a}$$

$$F_1 = \bar{c}_1 x_1 + \bar{c}_2 x_2^* - (\bar{b}_1 \bar{c}_2 + \bar{b}_2 \bar{c}_1)(x_1 + x_2^*), \tag{11b}$$

$$G_1 = 1 - \bar{b}_1 - \bar{b}_2 + (\bar{b}_1 \bar{b}_2 - \bar{c}_1 \bar{c}_2)(x_1 + x_2^*), \tag{11c}$$

$$H_1 = \bar{c}_1 + \bar{c}_2 - (\bar{b}_1 \bar{c}_2 + \bar{b}_2 \bar{c}_1)(x_1 + x_2^*), \tag{11d}$$

where $\bar{c}_1 = \eta_1 \beta_n^0 x_1$ and $\bar{c}_2 = \eta_1 \beta_n^0 x_2^*$ represent dimensionless damping coefficient groups, while $\bar{b}_1 = \chi_1 \beta_n^0 x_1$ and $\bar{b}_2 = \chi_2 \beta_n^0 x_2^*$ represent dimensionless inertial mass groups.

The complex eigen-frequency corresponding to the wavenumber β_n is denoted as ω_n . The n^{th} supplemental modal damping ratio of a cable ξ_n can be obtained by [17]:

$$\xi_n = \frac{\text{Im}[\omega_n]}{|\omega_n|} = \frac{\text{Im}[\beta_n]}{|\beta_n|} \cong \frac{\text{Im}[\Delta\beta_n]}{|\beta_n^0|}. \tag{12}$$

Substituting Equation (11) into Equation (12), the asymptotic supplemental modal damping ratio of the cable is finally derived as:

$$\xi_n \cong \frac{\bar{c}_1}{(1 - \bar{b}_1)^2 + (\bar{c}_1)^2} \frac{x_1}{L} + \frac{\bar{c}_2}{(1 - \bar{b}_2)^2 + (\bar{c}_2)^2} \frac{x_2}{L}. \tag{13}$$

If two identical IMDs are symmetrically installed at the cable for practical implementation, some simplifications in the notation can be introduced, i.e., $x = x_1 = L - x_2 = x_2^*$, $b_1 = b_2 = b$, $\chi_1 = \chi_2 = \chi$, $\bar{b}_1 = \bar{b}_2 = \bar{b}$, $c_1 = c_2 = c$, $\eta_1 = \eta_2 = \eta$, and $\bar{c}_1 = \bar{c}_2 = \bar{c}$. Equation (13) can be further simplified as:

$$\xi_n = \frac{2\bar{c}}{(1 - \bar{b})^2 + (\bar{c})^2} \frac{x_1}{L}. \tag{14}$$

3.3. Numerical Solution

The numerical solution of the wavenumber β_n to Equation (9) is obtained by the fixed point iteration [12,14], which starts from the undamped wavenumber β_n^0 . Substituting β_n^0 into the right side of the Equation (9), the resulting value β_n^1 is obtained. Similarly, with current estimate β_n^k , $k = 1, 2, 3, \dots$, a new estimate β_n^{k+1} will be derived. The iterative scheme is given by:

$$\beta_n^{k+1} L = n\pi + \arctan \frac{\bar{A}_1 + i\bar{B}_1}{\bar{C}_1 + i\bar{D}_1}, \tag{15}$$

$$\bar{A}_1 = -\chi_1 \sin^2 \beta_n^k x_1 - \chi_2 \sin^2 \beta_n^k x_2^* + (\chi_1 \chi_2 - \eta_1 \eta_2) \sin \beta_n^k x_1 \sin \beta_n^k x_2^* \sin \beta_n^k (x_1 + x_2^*), \quad (15a)$$

$$\bar{B}_1 = \eta_1 \sin^2 \beta_n^k x_1 + \eta_2 \sin^2 \beta_n^k x_2^* - (\chi_1 \eta_2 + \chi_2 \eta_1) \sin \beta_n^k x_1 \sin \beta_n^k x_2^* \sin \beta_n^k (x_1 + x_2^*), \quad (15b)$$

$$\begin{aligned} \bar{C}_1 = & 1 - \chi_1 \sin \beta_n^k x_1 \cos \beta_n^k x_1 - \chi_2 \sin \beta_n^k x_2^* \cos \beta_n^k x_2^* + \\ & (\chi_1 \chi_2 - \eta_1 \eta_2) \sin \beta_n^k x_1 \sin \beta_n^k x_2^* \cos \beta_n^k (x_1 + x_2^*) \end{aligned} \quad (15c)$$

$$\begin{aligned} \bar{D}_1 = & \eta_1 \sin \beta_n^k x_1 \cos \beta_n^k x_1 + \eta_2 \sin \beta_n^k x_2^* \cos \beta_n^k x_2^* - \\ & (\chi_1 \eta_2 + \chi_2 \eta_1) \sin \beta_n^k x_1 \sin \beta_n^k x_2^* \cos \beta_n^k (x_1 + x_2^*) \end{aligned} \quad (15d)$$

Finally, the supplemental modal damping ratio of a cable with two opposite IMDs can be calculated by Equation (12) after solving the wavenumber β_n .

3.4. Comparison of Asymptotic and Numerical Solutions

Figure 2 shows the comparison of asymptotic and numerical complex wavenumbers of a cable with two symmetric identical IMDs for various inertial masses. Two IMDs are assumed to be respectively installed at distances x_1 of 1% L and x_2 of 99% L from the left end of the cable, i.e., $x = x_1 = x_2^* = L - x_2 = 1\%L$. When inertial masses remain constant and damping coefficients of two IMDs increase from zero to infinity, the loci, which nearly trace a semicircular contour, start from the undamped wavenumber and finally attach to the real axis. According to Equation (12), the damping properties result from the imaginary part of the wavenumber. Maximum supplemental modal damping can be obtained at the top point of the semicircle [17]. The diameter of the loci is quite small but increases with the increase of the inertial masses of IMDs, indicating that two symmetric identical IMDs have slight influences on the damped frequency of the cable and can achieve higher supplemental modal damping ratios than traditional VDs. By comparing asymptotic and numerical complex wavenumbers, it is seen that two results coincide well with each other for small or moderate inertial mass ($\chi \leq 0.6 / (n\pi x / L)$) adopted in the IMDs. Nevertheless, the results deviate significantly from each other when the big inertial mass ($\chi = 0.9 / (n\pi x / L)$) shown in Figure 2d is adopted. Hence, solutions via numerical iteration are used to accurately predict the maximum supplemental modal damping ratio of the cable and corresponding optimal damper size of the IMD in the following discussions.

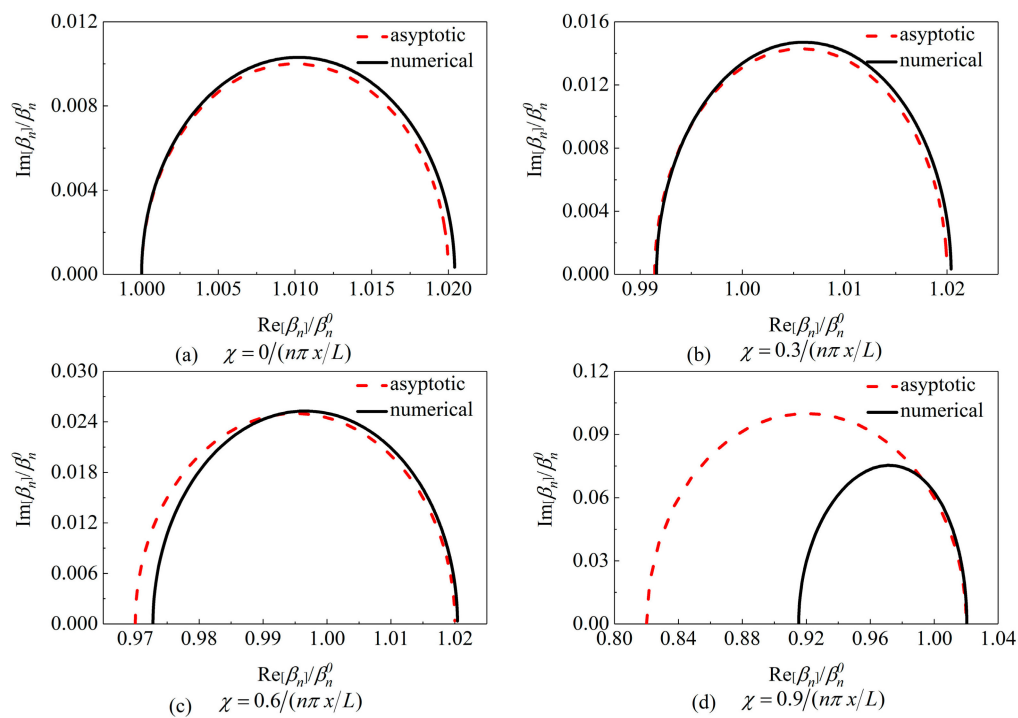


Figure 2. Comparison of asymptotic and numerical complex wavenumbers of a cable with two symmetric identical inertial mass dampers (IMDs) ($x = x_1 = x_2^* = 1\%L$).

3.5. Parametric Studies

Figure 3 presents the supplemental modal damping ratio of a cable with two symmetric identical VDs versus damping coefficients. For the convenience of comparisons, the results of a cable with a single VD are also shown. It is observed that symmetrically attaching two VDs on the opposite end of a cable is favorable to increasing the maximum supplemental damping ratio of the cable, and its maximum supplemental modal damping ratio is asymptotically the sum of contributions from each VD separately. The findings above are quite consistent with those reported in previous studies [76,77].

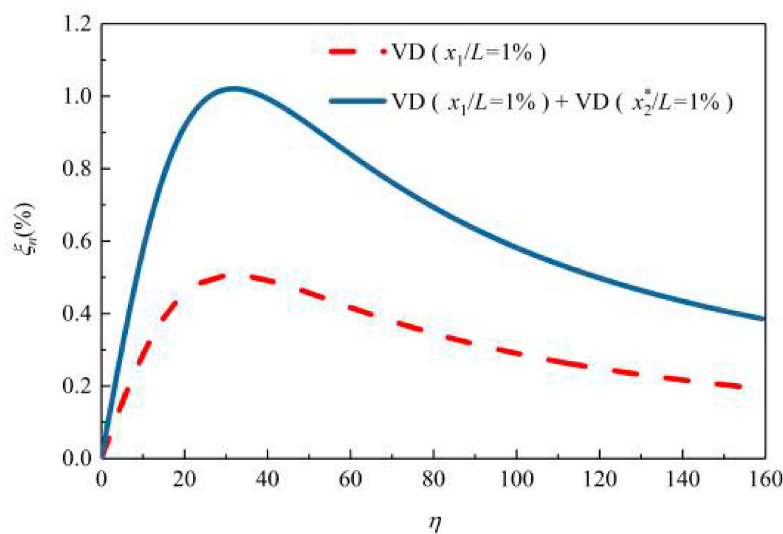


Figure 3. The supplemental modal damping ratio curve of a cable with a single viscous damper (VD) or two symmetric identical VDs ($x = x_1 = x_2^* = 1\%L$).

Figure 4 presents the supplemental modal damping ratio of a cable with two symmetric identical IMDs or a single IMD versus damping coefficients for various inertial masses. It is clear that two opposite IMDs can provide superior control performance to the cable over a single IMD. Figure 5 directly compares the maximum supplemental modal damping ratio of a cable equipped with a single IMD or two symmetric identical IMDs. For two symmetric identical IMDs with small or medium inertial mass, similarly to the case of two symmetric identical VDs, the maximum achievable supplemental damping ratio is approximately doubled with that provided by a single IMD. It indicates that two opposite IMDs are almost independent from each other. This finding may explain why the optimal damping coefficients of the IMDs for both the single configuration and two-symmetric configuration are similar to each other in magnitude, as shown in Figure 4. Moreover, the maximum achievable supplemental modal damping ratio of a cable provided by two symmetric identical IMDs is larger than that provided by a single IMD or two symmetric identical VDs.

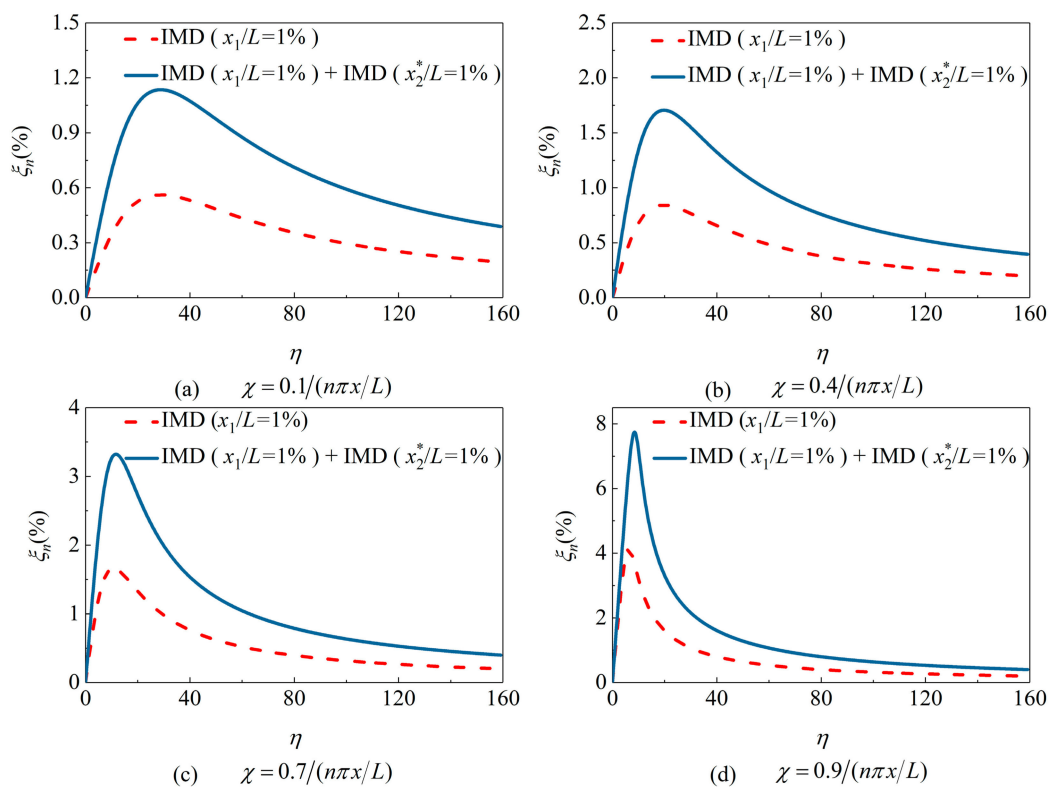


Figure 4. The supplemental modal damping ratio curve of a cable with a single IMD or two symmetric identical IMDs ($x = x_1 = x_2^* = 1\%L$).

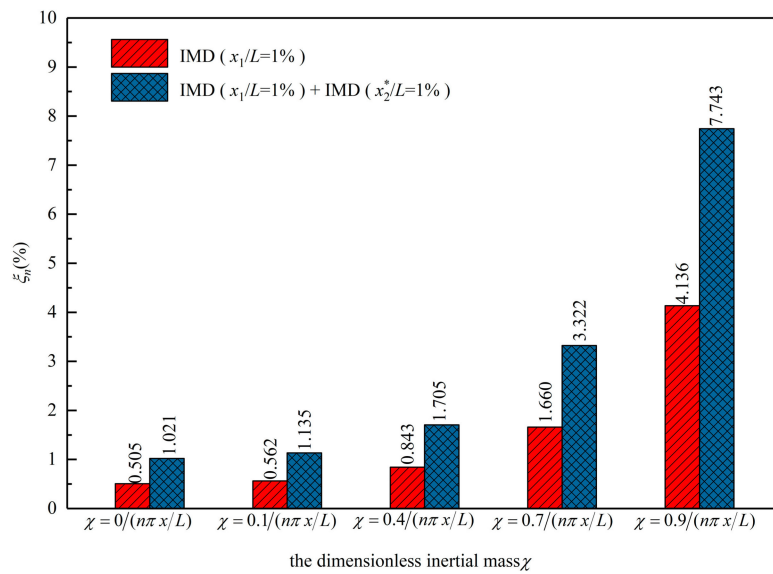


Figure 5. The maximum achievable supplemental modal damping ratio of a cable equipped with a single IMD or two symmetric identical IMDs ($x = x_1 = x_2^* = 1\%L$).

4. Two IMDs at the Same End

4.1. The Wavenumber Equation

When two IMDs are installed at the same end of the cable, substituting Equation (8) into Equation (7) and rearranging terms gives the following expression relating to x_1 and x_2 :

$$\tan \beta L = \frac{A_2 + iB_2}{C_2 + iD_2}, \tag{16}$$

$$A_2 = -\chi_1 \sin^2 \beta x_1 - \chi_2 \sin^2 \beta x_2 + (\chi_1 \chi_2 - \eta_1 \eta_2) \sin \beta x_1 \sin \beta x_2 \sin \beta(x_2 - x_1), \tag{16a}$$

$$B_2 = \eta_1 \sin^2 \beta x_1 + \eta_2 \sin^2 \beta x_2 - (\chi_1 \eta_2 + \chi_2 \eta_1) \sin \beta x_1 \sin \beta x_2 \sin \beta(x_2 - x_1), \tag{16b}$$

$$C_2 = 1 - \chi_1 \sin \beta x_1 \cos \beta x_1 - \chi_2 \sin \beta x_2 \cos \beta x_2 + (\chi_1 \chi_2 - \eta_1 \eta_2) \sin \beta x_1 \cos \beta x_2 \sin \beta(x_2 - x_1) \tag{16c}$$

$$D_2 = \eta_1 \sin \beta x_1 \cos \beta x_1 + \eta_2 \sin \beta x_2 \cos \beta x_2 - (\chi_1 \eta_2 + \chi_2 \eta_1) \sin \beta x_1 \cos \beta x_2 \sin \beta(x_2 - x_1) \tag{16d}$$

4.2. Asymptotic Solution

Similarly to the case of two opposite IMDs, assuming two IMDs locations $x_1, x_2 \ll L$ and the wave number β_n of the damped cable to be small perturbations from β_n^0 , Equation (16) can be simplified as:

$$\beta_n L \cong n\pi + \beta_n^0 \frac{E_2 + iF_2}{G_2 + iH_2}, \tag{17}$$

$$E_2 = -\bar{b}_1 x_1 - \bar{b}_2 x_2 + (\bar{b}_1 \bar{b}_2 - \bar{c}_1 \bar{c}_2)(x_2 - x_1), \tag{17a}$$

$$F_2 = \bar{c}_1 x_1 + \bar{c}_2 x_2 - (\bar{b}_1 \bar{c}_2 + \bar{b}_2 \bar{c}_1)(x_2 - x_1), \tag{17b}$$

$$G_2 = 1 - \bar{b}_1 - \bar{b}_2 + (\bar{b}_1 \bar{b}_2 - \bar{c}_1 \bar{c}_2)(1 - x_1/x_2), \tag{17c}$$

$$H_2 = \bar{c}_1 + \bar{c}_2 - (\bar{b}_1 \bar{c}_2 + \bar{b}_2 \bar{c}_1)(1 - x_1/x_2). \tag{17d}$$

From Equations (12) and (17), the asymptotic modal damping ratio of a cable with two IMDs at the same end can be obtained as:

$$\xi_n \cong \frac{\text{Im}[\Delta\beta_n]}{|\beta_n^0|} = \frac{F_2G_2 - E_2H_2}{(G_2)^2 + (H_2)^2}. \tag{18}$$

4.3. Numerical Solution

Similarly to the case of two opposite IMDs, Equation (16) can be solved for the wavenumber using the fixed-point iteration. Starting from the undamped wavenumber β_n^0 , the iterative scheme is given by:

$$\beta_n^{k+1}L = n\pi + \arctan \frac{\bar{A}_2 + i\bar{B}_2}{\bar{C}_2 + i\bar{D}_2}, \tag{19}$$

$$\bar{A}_2 = -\chi_1 \sin^2 \beta_n^k x_1 - \chi_2 \sin^2 \beta_n^k x_2 + (\chi_1\chi_2 - \eta_1\eta_2) \sin \beta_n^k x_1 \sin \beta_n^k x_2 \sin \beta_n^k (x_2 - x_1), \tag{19a}$$

$$\bar{B}_2 = \eta_1 \sin^2 \beta_n^k x_1 + \eta_2 \sin^2 \beta_n^k x_2 - (\chi_1\eta_2 + \chi_2\eta_1) \sin \beta_n^k x_1 \sin \beta_n^k x_2 \sin \beta_n^k (x_2 - x_1), \tag{19b}$$

$$\bar{C}_2 = 1 - \chi_1 \sin \beta_n^k x_1 \cos \beta_n^k x_1 - \chi_2 \sin \beta_n^k x_2 \cos \beta_n^k x_2 + (\chi_1\chi_2 - \eta_1\eta_2) \sin \beta_n^k x_1 \cos \beta_n^k x_2 \sin \beta_n^k (x_2 - x_1) \tag{19c}$$

$$\bar{D}_2 = \eta_1 \sin \beta_n^k x_1 \cos \beta_n^k x_1 + \eta_2 \sin \beta_n^k x_2 \cos \beta_n^k x_2 - (\chi_1\eta_2 + \chi_2\eta_1) \sin \beta_n^k x_1 \cos \beta_n^k x_2 \sin \beta_n^k (x_2 - x_1) \tag{19d}$$

After solving numerically for the wavenumber, the supplemental modal damping ratio of a cable with two IMDs at the same end can be determined from Equation (12).

4.4. Comparison of Asymptotic and Numerical Solutions

Figure 6 shows the comparison of asymptotic and numerical complex wavenumbers of a cable with two IMDs at the same end for various inertial masses, where two IMDs are installed at distances x_1 of 1% L and x_2 of 2% L from the left end of the cable, i.e., $x_1 = 1\%L, x_2 = 2\%L$. Seeing that two IMDs are usually identical, some simplifications in the notation are introduced, i.e., $b_1 = b_2 = b, \chi_1 = \chi_2 = \chi, c_1 = c_2 = c,$ and $\eta_1 = \eta_2 = \eta$. Similar to the case of two opposite IMDs, the loci start from the undamped wavenumber along a semicircular contour and finally attach to the real axis when damping coefficients of the IMDs increase from zero to infinity, and the effect of two IMDs installed at the same end of the cable on the cable frequency is also not significant. For a cable with two IMDs at the same end, although the asymptotic complex wavenumber agrees well with the numerical solution when the small inertial mass ($\chi \leq 0.3/(n\pi x_2/L)$) is used, it will lose accuracy when moderate or large inertial mass ($0.6/(n\pi x_2/L) \leq \chi \leq 0.9/(n\pi x_2/L)$) is adopted. Compared to the case of two opposite IMDs, prediction accuracies of the asymptotic solution are found to be relatively poor when two IMDs are installed at the same end of the cable. Hence, numerical results are used for the following parametric study.

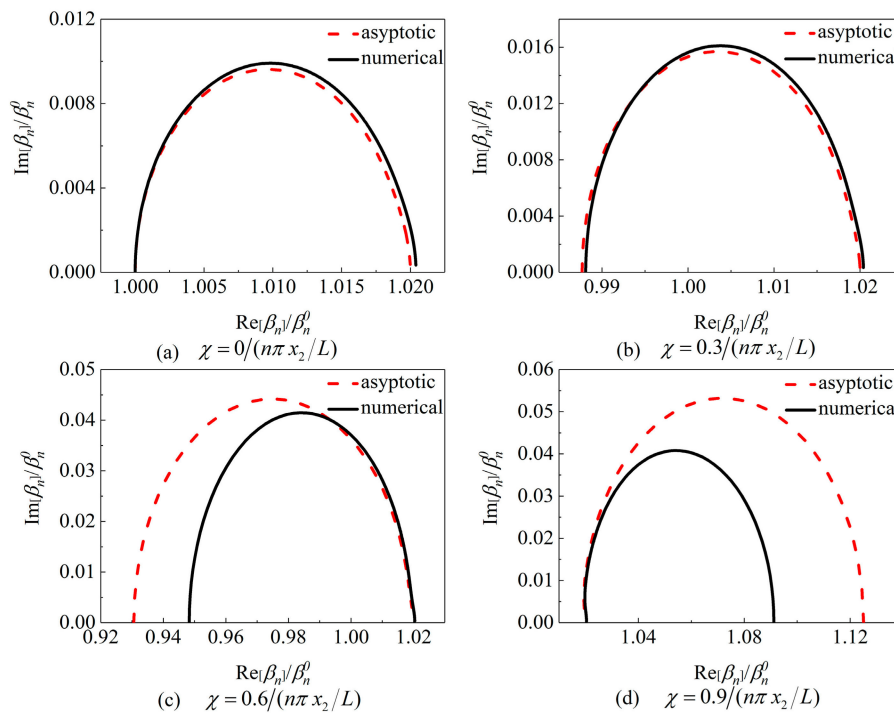


Figure 6. Comparison of asymptotic and numerical complex wavenumbers of a cable with two identical IMDs at the same end ($x_1 = 1\%L, x_2 = 2\%L$).

4.5. Parametric Studies

Figure 7 presents the supplemental modal damping ratio of a cable with two identical VDs at the same end or a single VD versus damping coefficients. It is observed that attaching two VDs at the same ends of the cable may help to reduce the damper size but cannot increase the maximum supplemental modal damping ratio. Moreover, its maximum modal damping ratio is slightly smaller than that provided by a single VD at the further distance. These observations are in agreement with previous findings [76,77].

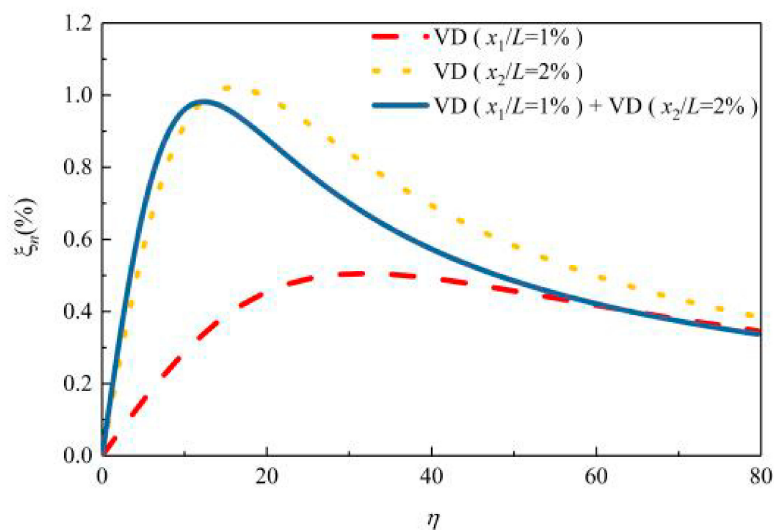


Figure 7. The modal damping ratios curves of a cable equipped with a single VD or two identical VDs at the same end ($x_1 = 1\%L, x_2 = 2\%L$).

Figure 8 presents the supplemental modal damping ratio of a cable with two identical IMDs at the same end or a single IMD versus damping coefficients of the IMD. If two IMDs with relatively small

or big inertial masses ($\chi \leq 0.1/(n\pi x_2/L)$ or $\chi = 0.9/(n\pi x_2/L)$) are installed at the same end of the cable, similarly to the case of two VDs at the same end of a cable, there is no advantage of increasing the maximum modal damping ratio over that of a single IMD. However, if moderate inertial mass ($0.4/(n\pi x_2/L) \leq \chi \leq 0.7/(n\pi x_2/L)$) of the IMD is used, it is interesting to observe that two IMDs at the same end can lead to smaller optimum damping coefficients and larger maximum supplemental modal damping ratios than that of a single IMD at a bigger distance.

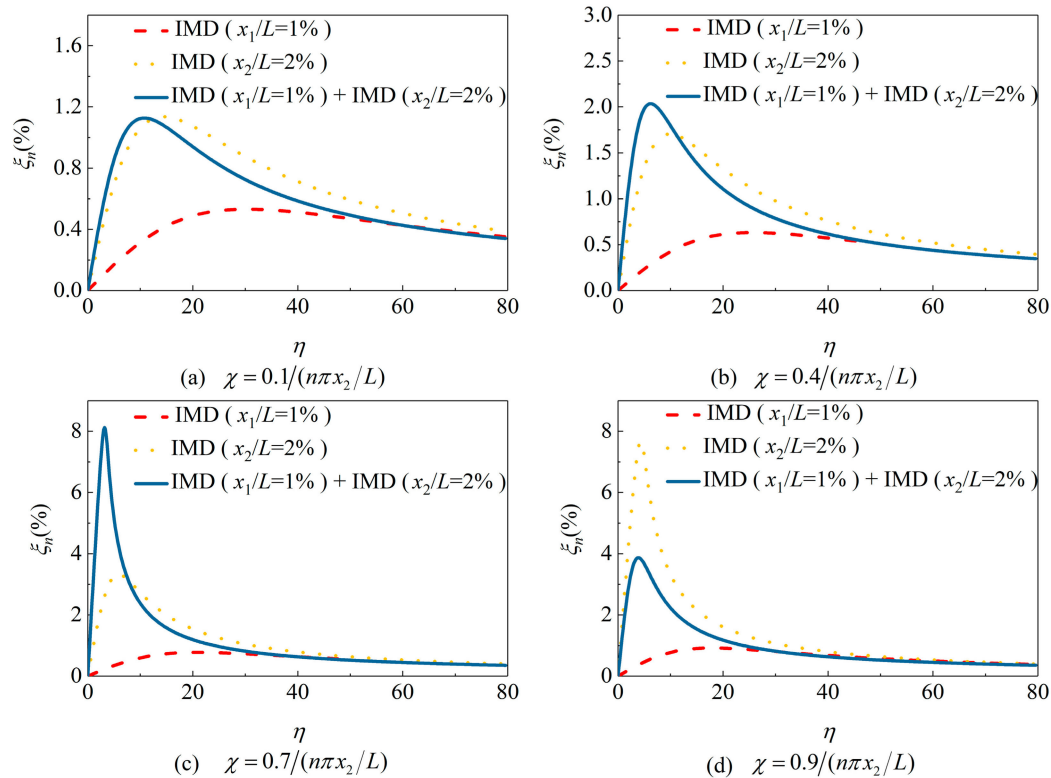


Figure 8. The modal damping ratios curves of a cable equipped with a single IMD or two identical IMDs at the same end ($x_1 = 1\%L, x_2 = 2\%L$).

The maximum achievable supplemental modal damping ratios of a cable equipped with a single IMD and two IMDs at the same end are directly compared in Figure 9. It is worth noting that the maximum supplemental modal damping ratio provided by two IMDs is higher than the sum of contributions from each IMD when inertial mass $\chi = 0.7/(n\pi x_2/L)$ is used. Though the strategy of two opposite IMDs has demonstrated that it can provide superior control performance, installing a damper at cable-tower anchorage is difficult and inconvenient. Thus, attaching two IMDs with appropriate inertial mass installed at the same end of the cable seems to be more promising for practical application.

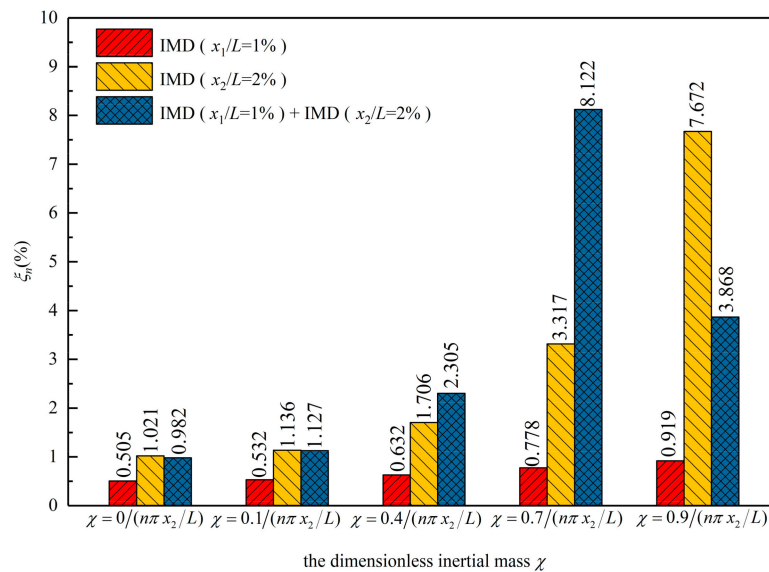


Figure 9. The maximum achievable supplemental modal damping ratio of a cable equipped with a single IMD or two identical IMDs at the same end ($x_1 = 1\%L$, $x_2 = 2\%L$).

5. Conclusions

In this paper, the combined damping effect of two discrete IMDs on a stay cable, either on the opposite end or the same end, was theoretically investigated in comparison with a single IMD, especially for the single-mode cable vibration control. Results showed that the maximum supplemental modal damping ratio of a cable provided by two opposite IMDs with small or moderate inertial mass is approximately the sum of contributions from each IMD. However, damping performances of the cable with two opposite IMDs will be reduced when the IMDs adopt relatively large inertial mass, in which the superposition effect of each IMD gets weak. As for a cable with two IMDs at the same end, the maximum modal damping ratio of the cable is smaller than that of a single IMD at the further distance when the IMDs adopt relatively small or large inertial mass. Fortunately, when the inertial mass of the IMD is appropriate, attaching two IMDs at the same end of the cable is able to obtain a larger maximum modal damping ratio than that of a single IMD at a bigger distance, which is even more than the sum of contributions from each IMD. Generally, attaching two opposite IMDs on a cable has shown better control performance than two IMDs at the same end. However, installing a damper at cable-tower anchorage is difficult and inconvenient. As an alternative, attaching two IMDs with appropriate inertial mass at the same end of a cable seems to be more promising for practical application. However, it is still necessary to explore the performance of two IMDs for the multimode cable vibration control, especially for super-long cables, which will be our consideration for further study.

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