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Predictive Control in Water Distribution Systems for Leak Reduction and Pressure Management via a Pressure Reducing Valve

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Abstract: This work proposes a model predictive control (MPC) strategy for pressure management and leakage reduction in a water distribution system (WDS). Unlike most of the reported models that mainly consider EPANET-based models, the proposed method considers its dynamic representation given by ordinary differential equations. The proposed MPC uses a pressure-reducing valve (PRV) as a control element to regulate the pressure in the WDS to track the demand. The control scheme proposes a strategy to manage the high nonlinearity of the PRV and takes into account the demand profile throughout the day as well as the leaks that occur in the pipeline. The estimates of magnitude and location of the leak are provided by an Extended Kalman Filter from previous work and with the aid of a rule-based set point manager reduces the fluid loss in the event of a leak. Different scenarios are studied to illustrate the effectiveness of the proposed control system, achieving an approximate reduction of up to 5% of water losses, demonstrating robustness in the case of uncertainty in the leak location estimate.

Keywords: pipelines; control valves; leak reduction; water distribution system; pressure management



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1. Introduction

Water distribution systems (WDS) are the most sustainable and efficient means of transporting fluids such as drinking water, natural gas, and oil [1]. These systems are composed of pipelines, pipe joints, connection nodes, and other components such as valves and pumps, which are prone to damage due to aging or unwanted events, such as earthquakes, floods, and lack of management, among others [2]. According to a study by the Organisation for Economic Co-operation and Development [3], these abnormal events cause water losses by leaks that reach almost 21%. The study examined water usage in 48 major cities across 17 countries, finding that in some cities of Mexico, the percentage was more than 40%. This highlights the need to propose technological developments to mitigate such losses.

From the control theory point of view, it is essential to investigate techniques to reduce the leaks without affecting the demand. In the literature, it has been found that the most effective approach to find a trade-off between maintaining the desired demand and reducing the water losses is by considering pressure water controllers on critical nodes of the water distribution network [4]. However, many challenges are associated with increasing demand and managing the pressure levels. In particular, pressure-reducing valves (PRV) are the recommended actuators to minimize these undesirable effects and operate the WDS effectively by following a pattern of demands [5].

Modeling the WDS is vital for designing any control algorithm [6,7]. These models are analyzed taking into account the location and number of PRVs for a correct evaluation of the pressure [8]. For instance, Mazumder et al. [9] developed an optimization method based on genetic algorithms for pressure management in a WDS by adjusting a PRV of a hydraulic network designed in EPANET. Parra et al. [10] proposed a pressure management system in EPANET composed of a PRV and a pump as a turbine where the hydraulic model played a decisive role identifying critical nodes and predicting hydraulics properties in the network. García-Ávila et al. [11] designed a water leakage minimization system by optimizing pressure using a PRV. The hydraulic model of the WDN was developed using EPANET and WaterNetGen software. Dini and Asadi [12] designed a methodology based on particle swarm optimization (PSO) to identify the PRV that requires adjustment and obtain pressure management in the system; the network model was designed in EPANET. Hernández et al. [13] proposed a detrended fluctuation analysis (or DFA) to highlight some of the traits such as the head loss of high-viscosity gas–liquid flows. Navarro et al. [14] designed a leak diagnosis system for a pipeline and residue analysis using genetic algorithms (GA) to minimize location error. Jara-Arriagada and Stoitianov [15] designed a sensitivity analysis system to evaluate the potential impact of pressure control in a WDS to reduce pipe breaks by applying a logistic regression technique. Moseithe et al. [16] proposed a review of techniques for locating PRVs and controlling pressure in a WDS, minimizing problems such as excessive pressure in the system. Mathye et al. [17] designed a pressure management system to reduce leaks through PRVs, taking into account consumption and leak flow. All those works developed algorithms for pressure management using hydraulic models based on EPANET and waterNetGen software, whose main limitation is that the models only work in steady-state behavior, while the effects of leaks and pressure changes are dynamic.

The present work proposes a dynamical approach to pressure management in a WDS. The transient effects due to pressure changes or leaks are modeled on the basis of water-hammer equations. Then, a constrained model predictive control (MPC) system is proposed to track the desired pressure profile driven by a set point manager that considers the water loss due to leaks and the demanded pressure profile. Moreover, a strategy to handle the high nonlinearity of the PRV control input is proposed. Finally, some simulations are proposed by considering the mathematical model of a real distribution system that can be configured as a single pipeline or a branched system. The results illustrate the effectiveness of the proposed method in the presence of physical constraints, noise, and transient behaviors due to leaks, saving up to 5% of water losses in the event of a leak and demonstrating robustness in the case of uncertainty in the leak location estimate. The rest of the document is organized as follows: Section 2 presents the considered case studies; Section 3 describes the problem formulation; Section 4 is devoted to the control strategy; Section 5 presents the simulation results; finally, Section 6 draws the conclusions.

2. Case Studies

Three different cases will be considered in this paper, corresponding to different hydraulic system structures with the pressure to be controlled at a specific position (called controlled node) and some leaks affecting the system at some other position.

2.1. Case 1: Pipeline with a Leak before the Controlled Node

In this case, the system is a pipeline under a leak like the one shown in Figure 1. It is composed of a reservoir that provides the fluid to the pipe divided into three sections for convenience. The first section is related to the distance between the inlet node and the leak (z_l); the second one is related to the distance from the leak to the controlled node ($z_2 = z_1 - z_l$); and finally, the third section is related to the distance between the controlled node and the outlet node (z_3).

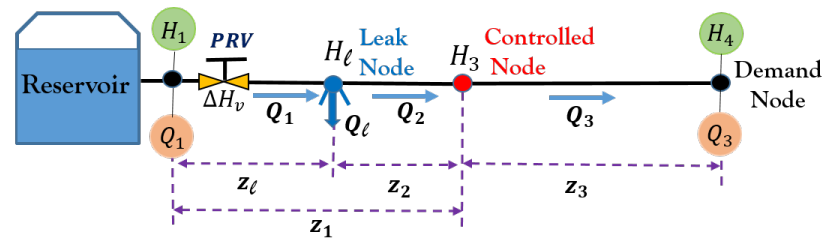


Figure 1. Case 1: Pipeline with a leak before the controlled node.

The mathematical model can be determined by considering water-hammer equations [18], which can be approximated on the basis of the three considered sections as follows (e.g., [19,20]):

$$\dot{Q}_1 = \frac{\alpha_1}{z_\ell} (H_1 - H_\ell) + \mu_1(Q_1)Q_1|Q_1| - \frac{\alpha_1}{z_1} \Delta H_v, \quad (1)$$

$$\dot{H}_\ell = \frac{\alpha_2}{z_\ell} (Q_1 - Q_2 - Q_\ell), \quad (2)$$

$$\dot{Q}_2 = \frac{\alpha_1}{z_2} (H_\ell - H_3) + \mu_2(Q_2)Q_2|Q_2|, \quad (3)$$

$$\dot{H}_3 = \frac{\alpha_2}{z_2} (Q_2 - Q_3), \quad (4)$$

$$\dot{Q}_3 = \frac{\alpha_1}{z_3} (H_3 - H_4) + \mu_3(Q_3)Q_3|Q_3|, \quad (5)$$

with

$$\alpha_1 = gA_r, \quad \alpha_2 = \frac{c^2}{gA_r}, \quad Q_\ell = \lambda\sqrt{|H_\ell|},$$

$$\mu_i(Q_i) = \frac{-f(Q_i)}{2DA_r}, \quad f(Q_i) = \frac{1.325}{\left[\ln\left(\frac{\epsilon}{3.7d} + \frac{5.74}{\left(\frac{4Q_i}{\pi d v}\right)^{0.9}} \right) \right]^2},$$

where H_1 , H_ℓ , H_3 , and H_4 are the piezometric heads (m) at the inlet, leak node, controlled node, and outlet, respectively; Q_1 , Q_2 , Q_3 are the volumetric flow rates in each section (m^3/s); g is the gravitational constant (m/s^2); A_r is the cross-sectional area of the pipe (m^2); c is the wave speed (m/s); d is the pipeline diameter (m); v represents the kinematic viscosity; the friction term is $f(Q_i)$; ϵ is the roughness of the pipe; and λ is the leak coefficient. Finally, ΔH_v describes the PRV effect.

A PRV is an actuator used to reduce downstream pressure. Internally, a PRV is made of a fixed orifice, a pilot valve, and a needle valve [21]. The shutter is the outer mechanism for adjusting the outlet pressure, either increasing or decreasing the pressure in a range from 0 to 100%. Figure 2 shows the schematic view of a PRV, where H_{v1} and H_{v2} are the piezometric head at valve ends, $r \in (0, 1]$ is the valve adjustment (control input), and Q_1 is the flow through the valve.

The differential pressure in a PRV is described as [22]:

$$\Delta H_v = H_{v1} - H_{v2} = \frac{Q_1|Q_1|}{(rE)^2}, \quad (6)$$

where $E = C_v A_v (2g)^{1/2}$ is the Torricelli expression, C_v is the discharge coefficient of the valve, and A_v is the cross-sectional area of the valve (m^2).

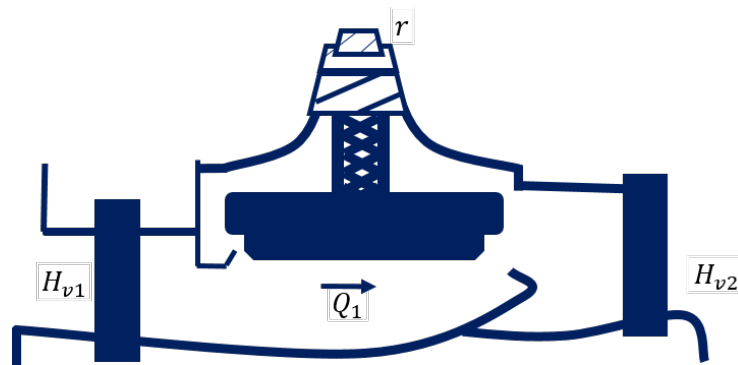


Figure 2. Schematic view of a PRV.

2.2. Case 2: Pipeline with a Leak after the Controlled Node

When the leak appears after the controlled node, as shown in Figure 3, then $z_2 = z_3 - z_\ell$, and the model becomes

$$\dot{Q}_1 = \frac{\alpha_1}{z_1} (H_1 - H_2) + \mu_1(Q_1)Q_1|Q_1| - \frac{\alpha_1}{z_1} \Delta H_v, \tag{7}$$

$$\dot{H}_2 = \frac{\alpha_2}{z_1} (Q_1 - Q_2), \tag{8}$$

$$\dot{Q}_2 = \frac{\alpha_1}{z_\ell} (H_2 - H_3) + \mu_2(Q_2)Q_2|Q_2|, \tag{9}$$

$$\dot{H}_\ell = \frac{\alpha_2}{z_\ell} (Q_2 - Q_3 - \lambda\sqrt{H_\ell}), \tag{10}$$

$$\dot{Q}_3 = \frac{\alpha_1}{z_2} (H_3 - H_4) + \mu_3(Q_3)Q_3|Q_3|. \tag{11}$$

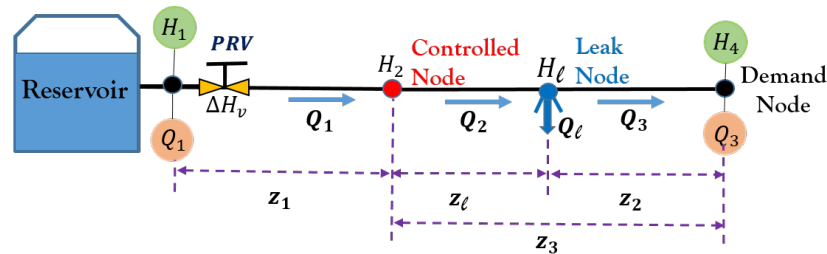


Figure 3. Case 2: Pipeline with a leak after the controlled node.

2.3. Case 3: Branched Water Distribution Network

In this third case, a branched water distribution network is considered with the topology shown in Figure 4. With the same notations as before, the mathematical model describing branch flows Q_i and node pressures H_i can be given by:

$$\dot{Q}_1(t) = \frac{\alpha_1}{z_1} (H_{B1} - H_2) + \mu_1(Q_1)Q_1|Q_1| - \Delta H_v, \tag{12}$$

$$\dot{H}_2(t) = \frac{\alpha_2}{z_1} (Q_1 - Q_2 - Q_4), \tag{13}$$

$$\dot{Q}_2(t) = \frac{\alpha_1}{z_2} (H_2 - H_\ell) + \mu_2(Q_2)Q_2|Q_2|, \tag{14}$$

$$\dot{H}_\ell(t) = \frac{\alpha_2}{z_2} (Q_2 - Q_5 - Q_3 - Q_\ell), \tag{15}$$

$$\dot{Q}_3(t) = \frac{\alpha_1}{z_3}(H_3 - H_{B2}) + \mu_3(Q_3)Q_3|Q_3|, \quad (16)$$

$$\dot{Q}_4(t) = \frac{\alpha_1}{z_4}(H_2 - H_{B3}) + \mu_4(Q_4)Q_4|Q_4|, \quad (17)$$

$$\dot{Q}_5(t) = \frac{\alpha_1}{z_5}(H_3 - H_{B4}) + \mu_5(Q_5)Q_5|Q_5|. \quad (18)$$

The mathematical models developed here are valid for any pipeline system with the configurations described in Figures 1–4. For validation tests, the physical parameters were taken from a real system located at the Hydroinformatics Laboratory of the Technological Institute of Tuxtla Gutiérrez, whose mathematical model was presented in [23].

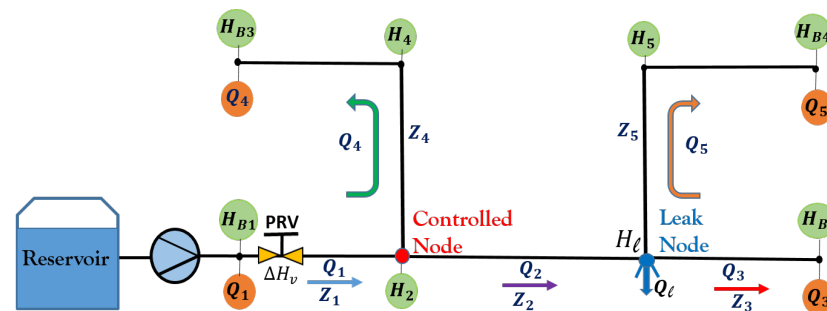


Figure 4. Case 3: Branched water distribution network.

Remark 1. Notice that λ and z_ℓ are unknown parameters that must be estimated by considering a leak detection and localization method. For Cases 1 and 2, we consider our previous result published in [24] where a leak location and estimation method was designed using an extended Kalman filter (EKF). For Case 3, an EKF as the one in [25] can be used. Therefore, for the sake of simplicity, λ and z_ℓ are assumed to be known.

3. Problem Formulation

WDS are designed to meet the desired demands at their ends, even when affected by leaks. It is important to note that these demands have different profiles depending on the time and day. Typically, the demand is higher during the day and lower during the early morning [26]. In case of a leak event, the fluid loss rate (leak magnitude) can be reduced by reducing the pressure on the controlled node. However, this also reduces the flow at the node of demand, compromising the main objective of the WDS. In this regard, the control strategy for reducing leaks must consider a time-varying profile, with the primary objective to maintain a trade-off between reducing the leak magnitude and maintaining the desired demand.

To address this problem, the control scheme shown in Figure 5 is proposed. This scheme is made of three components: an extended Kalman filter (EKF), a pressure controller, and a set point manager. The EKF is used to estimate the flows and pressures along the WDS and to detect and estimate the leak position and its magnitude by using only pressure head and flow rate measurements at the pipeline ends. The EKF considered for Cases 1 and 2 is the one reported in our previous work in [24] and, for Case 3, the one in [27] is considered; however, any other leak location and estimation method can be used for the proposed scheme. The pressure controller is an MPC that takes into account physical constraints and the leak dynamics. For its operation, the MPC uses the estimated values obtained from the EKF ($\hat{z}_\ell, \hat{\lambda}, \hat{H}_\ell$) and measured flows and pressures from the WDS. It is important to mention that the MPC also considers the PRV model whose behavior is highly nonlinear due to the inverse quadratic term of its control variable (r), which represents a challenge for the control system. Finally, the set point manager block provides the reference

pressure (s_k) to the MPC; it handles the trade-off between fluid loss and fulfilment of a demanded pressure profile (DP) in the event of a leak.

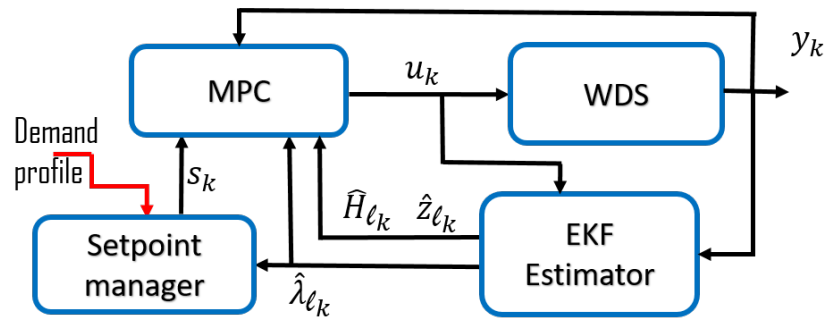


Figure 5. Block diagram representation of MPC used in WDS.

4. Pressure Control

The control strategy adopted to regulate the pressure in the WDS is an MPC. Figure 6 shows the implemented scheme which will be detailed in this section. The basic idea in MPC is to calculate the control action at each sampling instant through the solution of an optimization problem, which is written in terms of a prediction model.

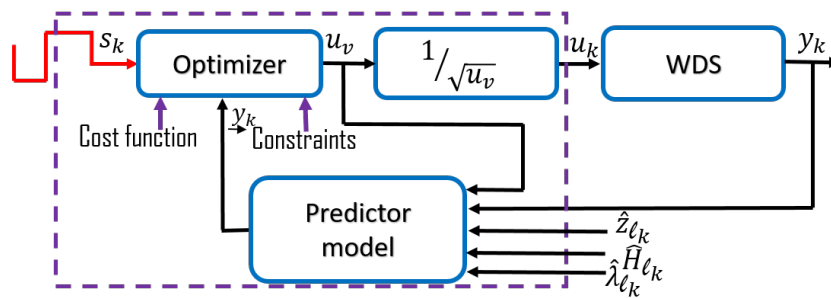


Figure 6. MPC scheme for pressure control.

4.1. PRV Handling

The PRV has the function of regulating the incoming water for a safer constant pre-determined downstream level. The control signal r establishes the downstream pressure. One of the main challenges in using a PRV is its high nonlinearity as an inverse quadratic term (r) represents its control variable. This term affects the control input considerably because the more the PRV is closed, the greater the effects on the pressure. Then, to use linear models in the MPC formulation, the PRV control variable r is substituted in (6) by a virtual input u_v to artificially hide the nonlinear behavior of the PRV for the prediction model. This is accomplished with:

$$\Delta H_v = \frac{Q_1|Q_1|u_v}{E^2}, \quad \text{with } u_v = \frac{1}{r^2}, \tag{19}$$

where $u_v \in [1, \infty)$, but for practical reasons u_v ranges from 1 (for the PRV fully open, $r = 1$) to 100 (for the PRV 90% closed, $r = 0.1$). This new term is substituted in the three models discussed in Section 2. In this way, the MPC calculates u_v instead of r . Next, u_v is transformed back to r before it is applied to the WDS with $u = r = \frac{1}{\sqrt{u_v}}$, as shown in Figure 6.

4.2. Prediction Models

The standard MPC strategy considers linear prediction models [28,29], so the models of the case studies listed before need to be linearized.

For a pipeline with a leak before the controlled node, the state and input vectors are defined as $\bar{x} = [x_1, x_2, x_3, x_4, x_5] = [Q_1, H_\ell, Q_2, H_3, Q_3]$, and $\bar{u} = [H_1, H_4, \lambda, u_v]^T$. The system (1)–(5) is linearized at the operating point (\bar{x}^*, \bar{u}^*) taking the form $\dot{\bar{x}}(t) = A_c \bar{x}(t) + B_c \bar{u}(t) + \delta_c$, where δ_c is the offset caused by linearization. The Jacobian matrices are given by:

$$A_c = \begin{bmatrix} a_{11} & -\frac{\alpha_1}{z_\ell} & 0 & 0 & 0 \\ \frac{\alpha_2}{z_\ell} & 0 & -\frac{\alpha_2}{z_\ell} & 0 & 0 \\ 0 & \frac{\alpha_1}{z_2} & a_{33} & -\frac{\alpha_1}{z_2} & 0 \\ 0 & 0 & \frac{\alpha_2}{z_2} & 0 & -\frac{\alpha_2}{z_2} \\ 0 & 0 & 0 & \frac{\alpha_1}{z_3} & a_{55} \end{bmatrix},$$

$$B_c = \begin{bmatrix} \frac{\alpha_1}{z_\ell} & 0 & 0 & \frac{-gA_r x_1^* |x_1^*|}{E^2 z_\ell} \\ 0 & 0 & -\frac{\alpha_2 \sqrt{H_2^*}}{z_2} & 0 \\ 0 & 0 & 0 & 0 \\ 0 & 0 & 0 & 0 \\ 0 & \frac{-\alpha_1}{z_3} & 0 & 0 \end{bmatrix},$$

with

$$a_{11} = 2|x_1^*| \mu(x_1^*) + \frac{2(1.325)}{0.9 \frac{5.74}{(\frac{4}{\pi d v})^{0.9}}} \frac{(x_1^*)^{1.9}}{\ln \left(\left(\frac{\epsilon}{3.7d} \right) + \frac{5.74}{(\frac{4x_1^*}{\pi d v})^{0.9}} \right)^3} - \frac{2A_r g |x_1^*| u_v^*}{z_1 E^2},$$

$$a_{33} = 2|x_3^*| \mu(x_3^*) + \frac{2(1.325)}{0.9 \frac{5.74}{(\frac{4}{\pi d v})^{0.9}}} \frac{(x_3^*)^{1.9}}{\ln \left(\left(\frac{\epsilon}{3.7d} \right) + \frac{5.74}{(\frac{4x_3^*}{\pi d v})^{0.9}} \right)^3},$$

$$a_{55} = 2|x_5^*| \mu(x_5^*) + \frac{2(1.135)}{0.9 \frac{5.74}{(\frac{4}{\pi d v})^{0.9}}} \frac{(x_5^*)^{1.9}}{\ln \left(\left(\frac{\epsilon}{3.7d} \right) + \frac{5.74}{(\frac{4x_5^*}{\pi d v})^{0.9}} \right)^3},$$

$$\delta_c = f_1(\bar{x}^*, \bar{u}^*) - A_c \bar{x}^* - B_c \bar{u}^* + \Delta_1,$$

where $f_1(\bar{x}^*, \bar{u}^*)$ is the nonlinear model of Case 1 evaluated at the linearization point, and Δ_1 gathers terms of order larger than 1.

For a pipeline with a leak after the controlled node, the state and input vectors are defined as $\bar{x} = [x_1, x_2, x_3, x_4, x_5] = [Q_1, H_2, Q_2, H_\ell, Q_3]$, $\bar{u} = [H_1, H_4, \lambda, u_v]^T$. Using a_{11} , a_{33} , a_{55} from the previous case, the Jacobian matrices for system (7)–(11) are:

$$A_c = \begin{bmatrix} a_{11} & -\frac{\alpha_1}{z_1} & 0 & 0 & 0 \\ \frac{\alpha_2}{z_1} & 0 & -\frac{\alpha_2}{z_1} & 0 & 0 \\ 0 & \frac{\alpha_1}{z_\ell} & a_{33} & -\frac{\alpha_1}{z_\ell} & 0 \\ 0 & 0 & \frac{\alpha_2}{z_\ell} & 0 & \frac{\alpha_2}{z_\ell} \\ 0 & 0 & 0 & \frac{\alpha_1}{z_2} & a_{55} \end{bmatrix},$$

$$B_c = \begin{bmatrix} \frac{a_1}{z_1} & 0 & 0 & \frac{-gA_r x_1^* |x_1^*|}{E^2 z_1} \\ 0 & 0 & 0 & 0 \\ 0 & 0 & 0 & 0 \\ 0 & 0 & -\frac{a_2 \sqrt{H_4^*}}{z_\ell} & 0 \\ 0 & \frac{-a_1}{z_2} & 0 & 0 \end{bmatrix},$$

$$\delta_c = f_2(\bar{x}^*, \bar{u}^*) - A_c \bar{x}^* - B_c \bar{u}^* + \Delta_2,$$

where $f_2(\bar{x}^*, \bar{u}^*)$ and Δ_2 refer to the nonlinear model of Case 2 as before.

For the branched water distribution network, the states and the inputs are defined as $\bar{x} = [x_1, x_2, x_3, x_4, x_5, x_6, x_7] = [Q_1, H_2, Q_2, H_\ell, Q_3, Q_4, Q_5]$, $\bar{u} = [H_{B1}, H_{B2}, H_{B3}, H_{B4}, \lambda, u_v]^T$. The Jacobian matrices for system (12)–(18) are:

$$A_c = \begin{bmatrix} a_{11} & -\frac{\alpha_1}{z_1} & 0 & 0 & 0 & 0 & 0 \\ \frac{\alpha_2}{z_1} & 0 & -\frac{\alpha_2}{z_2} & 0 & 0 & -\frac{\alpha_2}{z_1} & 0 \\ 0 & \frac{\alpha_1}{z_2} & a_{33} & -\frac{\alpha_1}{z_2} & 0 & 0 & 0 \\ 0 & 0 & \frac{\alpha_2}{z_2} & 0 & -\frac{\alpha_2}{z_2} & 0 & -\frac{\alpha_2}{z_2} \\ 0 & 0 & 0 & \frac{\alpha_1}{z_3} & a_{55} & 0 & 0 \\ 0 & \frac{\alpha_1}{z_4} & 0 & 0 & 0 & a_{66} & 0 \\ 0 & 0 & 0 & \frac{\alpha_1}{z_5} & 0 & 0 & a_{77} \end{bmatrix},$$

$$B_c = \begin{bmatrix} \frac{\alpha_1}{z_1} & 0 & 0 & 0 & 0 & \frac{-gA_r x_1^* |x_1^*|}{E^2 z_1} \\ 0 & 0 & 0 & 0 & 0 & 0 \\ 0 & 0 & 0 & 0 & 0 & 0 \\ 0 & 0 & 0 & 0 & \frac{-a_2 \sqrt{H_3^*}}{z_1} & 0 \\ 0 & \frac{-\alpha_1}{z_3} & 0 & 0 & 0 & 0 \\ 0 & 0 & \frac{-\alpha_1}{z_4} & 0 & 0 & 0 \\ 0 & 0 & 0 & \frac{-\alpha_1}{z_5} & 0 & 0 \end{bmatrix},$$

with

$$a_{11} = 2|x_1^*| \mu(x_1^*) + \frac{2(1.135)}{0.9 \frac{5.74}{(\frac{4}{\pi d v})^{0.9}}} \frac{(x_1^*)^{1.9}}{\ln\left(\left(\frac{\epsilon}{3.7d}\right) + \frac{5.74}{(\frac{4x_1^*}{\pi d v})^{0.9}}\right)^3} - \frac{2A_r g |x_1^*| u_v^*}{z_1 E^2},$$

$$a_{33} = 2|x_3^*| \mu(x_3^*) + \frac{2(1.135)}{0.9 \frac{5.74}{(\frac{4}{\pi d v})^{0.9}}} \frac{(x_3^*)^{1.9}}{\ln\left(\left(\frac{\epsilon}{3.7d}\right) + \frac{5.74}{(\frac{4x_3^*}{\pi d v})^{0.9}}\right)^3},$$

$$a_{55} = 2|x_5^*| \mu(x_5^*) + \frac{2(1.135)}{0.9 \frac{5.74}{(\frac{4}{\pi d v})^{0.9}}} \frac{(x_5^*)^{1.9}}{\ln\left(\left(\frac{\epsilon}{3.7d}\right) + \frac{5.74}{(\frac{4x_5^*}{\pi d v})^{0.9}}\right)^3},$$

$$a_{66} = 2|x_6^*| \mu(x_6^*) + \frac{2(1.135)}{0.9 \frac{5.74}{(\frac{4}{\pi d v})^{0.9}}} \frac{(x_6^*)^{1.9}}{\ln\left(\left(\frac{\epsilon}{3.7d}\right) + \frac{5.74}{(\frac{4x_6^*}{\pi d v})^{0.9}}\right)^3},$$

$$a_{77} = 2|x_3^*|\mu(x_7^*) + \frac{2(1.135)}{0.9 \frac{5.74}{(\frac{4}{\pi d v})^{0.9}}} \frac{(x_7^*)^{1.9}}{\ln\left(\left(\frac{\epsilon}{3.7d}\right) + \frac{5.74}{(\frac{4x_7^*}{\pi d v})^{0.9}}\right)^3},$$

$$\delta_c = f_3(\bar{x}^*, \bar{u}^*) - A_c \bar{x}^* - B_c \bar{u}^* + \Delta_3,$$

where $f_3(\bar{x}^*, \bar{u}^*)$ and Δ_3 refer to the nonlinear model of Case 3.

For all cases, the controlled output is $y(t) = C_c \bar{x}(t) = x_2(t)$. It is also noted that the only manipulated input is u_v . The rest of the elements in \bar{u} are non-manipulated in the three cases.

Now for a discrete-time implementation, the previous linear models are discretized, with a sample time T_s , leading to systems of the form

$$x_{k+1} = A_d x_k + B_d u_k + \delta_d; \quad y_k = C_d x_k. \tag{20}$$

The MPC calculates control input increments; then each model is augmented to express it in those terms:

$$\zeta_{k+1} = \underbrace{\begin{bmatrix} A_d & B_d \\ 0 & I \end{bmatrix}}_A \zeta_k + \underbrace{\begin{bmatrix} B_d \\ I \end{bmatrix}}_B \Delta u_k + \underbrace{\begin{bmatrix} \delta_d \\ 0 \end{bmatrix}}_\delta, \tag{21}$$

$$y_k = \underbrace{\begin{bmatrix} C_d & 0 \end{bmatrix}}_C \zeta_k, \tag{22}$$

where $\zeta = [x_k^T \ u_{k-1}^T]^T$ and $\Delta u_k = u_k - u_{k-1}$.

4.3. Prediction Equations

The predictions equations of the augmented-state, output and input over a prediction horizon n_y , and control horizon n_u are:

$$\underline{\zeta}_k = P_{\zeta\zeta} \zeta_k + P_{\zeta\Delta u} \underline{\Delta u}_{k-1} + P_{\zeta\delta} \delta, \tag{23}$$

$$\underline{y}_k = P_{y\zeta} \zeta_k + P_{y\Delta u} \underline{\Delta u}_{k-1} + P_{y\delta} \delta, \tag{24}$$

$$\underline{u}_{k-1} = P_{u\Delta u} \underline{\Delta u}_{k-1} + P_{u\zeta} \zeta_k, \tag{25}$$

where the arrow notation denotes prediction and is defined as $\underline{x}_k = [x_{k+1}^T \ x_{k+2}^T \ \dots]^T$.

Augmented state prediction matrices are

$$P_{\zeta\zeta} = \begin{bmatrix} A \\ A^2 \\ \vdots \\ A^{n_y} \end{bmatrix}, \quad P_{\zeta\Delta u} = \begin{bmatrix} B & \dots & 0 \\ AB & \dots & 0 \\ \vdots & \ddots & \vdots \\ A^{n_y-1}B & \dots & A^{n_y-n_u}B \end{bmatrix}, \quad P_{\zeta\delta} = \begin{bmatrix} I \\ I + A \\ \vdots \\ I + \sum_{i=1}^{n_y-1} A^i \end{bmatrix}.$$

Output prediction matrices are $P_{y\zeta} = \text{diag}(C)P_{\zeta\zeta}$, $P_{y\Delta u} = \text{diag}(C)P_{\zeta\Delta u}$, $P_{y\delta} = \text{diag}(C)P_{\zeta\delta}$. Input predictions matrices are given by:

$$\underline{u}_{k-1} = \underbrace{\begin{bmatrix} I & 0 & \dots & 0 \\ I & I & \dots & 0 \\ \vdots & \vdots & \ddots & \vdots \\ I & I & \dots & I \end{bmatrix}}_{P_{u\Delta u}} \underline{\Delta u}_{k-1} + \underbrace{\text{col}\left(\begin{bmatrix} 0 & I \end{bmatrix}\right)}_{P_{u\zeta}} \zeta_k, \tag{26}$$

and for the states $\underline{x}_{\gamma k} = P_{x\zeta} \zeta_{\gamma k} = \text{diag}(\begin{bmatrix} I & 0 \end{bmatrix}) \zeta_{\gamma k}$.

To construct the prediction vectors, full state availability is required. Although $H_\ell(t)$ is not a measured state and λ, z_ℓ are unknown parameters, all are considered known as they could be provided by the EKF estimator. The rest of the states are measured flows and pressures.

4.4. Cost Function and Constraints

The cost function to be optimized penalizes the future output error with respect to the desired output value s_k and control input along the prediction and control horizons:

$$J = \sum_{i=1}^{n_y} (y_{k+i} - s_k)^T Q (y_{k+i} - s_k) + \sum_{i=0}^{n_u-1} \Delta u_{k+i}^T \mathcal{R} \Delta u_{k+i} \quad (27)$$

with $Q > 0$ and $\mathcal{R} > 0$.

Constraints of the form:

$$\begin{aligned} \text{diag}(A_x) \underline{x}_{\gamma k} &\leq \text{col}(b_x); \quad \text{diag}(A_y) \underline{y}_{\gamma k} \leq \text{col}(b_y); \\ \text{diag}(A_u) \underline{u}_{\gamma k-1} &\leq \text{col}(b_u); \quad \text{diag}(A_{\Delta u}) \underline{\Delta u}_{\gamma k-1} \leq \text{col}(b_{\Delta u}) \end{aligned} \quad (28)$$

are considered, where $A_x, A_y, A_u, A_{\Delta u} = [I, -I]^T$ and $b_x, b_y, b_u, b_{\Delta u}$ are vectors that contain the maximum and minimum values allowed in the form $[\max \ \min]^T$.

4.5. Control Law

Cost function (27) and constraints (28) can be expressed in terms of the decision variable Δu [30], and the following optimization problem is obtained:

$$\begin{aligned} \underline{\Delta u}_{\gamma k-1}^* &= \arg \min_{\underline{\Delta u}_{\gamma k-1}} \left\{ \frac{1}{2} \underline{\Delta u}_{\gamma k-1}^T \mathcal{H} \underline{\Delta u}_{\gamma k-1} + \mathcal{F}^T \underline{\Delta u}_{\gamma k-1} \right\} \\ \text{s.t. } & M_c \zeta_k + N_c \underline{\Delta u}_{\gamma k-1} \leq f_c, \end{aligned}$$

with

$$\begin{aligned} \mathcal{H} &= P_{y\Delta u}^T \text{diag}(Q) P_{y\Delta u} + \text{diag}(\mathcal{R}); \\ \mathcal{F} &= P_{y\Delta u} \text{diag}(Q) (P_{yx} \zeta_k + P_{y\delta} \delta - \hat{\underline{s}}_{\gamma k}); \\ M_c &= \begin{bmatrix} \text{diag}(A_x) P_{x\zeta} P_{\zeta\zeta} \\ \text{diag}(A_y) P_{y\zeta} \\ \text{diag}(A_u) P_{u\zeta} \\ 0 \end{bmatrix}; \\ N_c &= \begin{bmatrix} \text{diag}(A_x) P_{x\zeta} P_{\zeta\Delta u} \\ \text{diag}(A_y) P_{y\Delta u} \\ \text{diag}(A_u) P_{u\Delta u} \\ \text{diag}(A_{\Delta u}) \end{bmatrix}; \\ f_c &= \begin{bmatrix} \text{col}(b_x) \\ \text{col}(b_y) \\ \text{col}(b_u) \\ \text{col}(b_{\Delta u}) \end{bmatrix} - \begin{bmatrix} \text{diag}(A_x) P_{x\zeta} P_{\zeta\delta} \\ \text{diag}(A_y) P_{y\delta} \\ 0 \\ 0 \end{bmatrix} \delta. \end{aligned}$$

Finally, the control law is given by

$$u_k = u_{k-1} + \Delta u_k^*, \quad (29)$$

where Δu_k^* is the first element of $\underline{\Delta u}_{\gamma k-1}^*$.

4.6. Offset-Free Tracking

The MPC presented earlier does not contain an explicit mechanism to deal with disturbances and modeling errors that arise from set point changes that move the system away from the original operation point where it was linearized. Therefore, an integral action must be embedded into the control law to deal with these situations and guarantee zero steady-state error. This is performed including in the state space model an integrating state

$$\begin{bmatrix} \dot{\bar{x}}(t) \\ \dot{\bar{\zeta}}(t) \end{bmatrix} = \begin{bmatrix} A_c & 0 \\ -C_c & 0 \end{bmatrix} \begin{bmatrix} \bar{x}(t) \\ \bar{\zeta}(t) \end{bmatrix} + \begin{bmatrix} B_c \\ 0 \end{bmatrix} \bar{u}(t) + \begin{bmatrix} \delta_c \\ s(t) \end{bmatrix},$$

$$y(t) = \begin{bmatrix} C_c & 0 \end{bmatrix} \begin{bmatrix} \bar{x}(t) \\ \bar{\zeta}(t) \end{bmatrix}.$$

A stabilizing feedback gain can be computed to guarantee that in steady-state $y(t) = x_2(t) = s(t)$. Another approach to obtain integral action could be to estimate the disturbance with the use of a disturbance observer [31].

4.7. Set Point Manager

The set point manager (SPM) is a ruled-based algorithm that takes into account the periods of expected maximum and minimum demand through time conditions, that is, it identifies the hours (early morning) in which it is possible to reduce the pressure to reduce the leak magnitude without affecting the demand. Therefore, when the system is leak-free (i.e., $\lambda = 0$), the SPM computes the pressure reference (s_k) by solving the hydraulic model to ensure that the demand is satisfied. Then, if there is no fluid loss, the priority is to supply to the WDS the pressure needed to fulfil the users' demand in both maximum and minimum demand hours.

On the other hand, the SPM receives the leak magnitude estimate through the leak coefficient λ_k when a leak occurs. The SPM identifies if it is possible to reduce the pressure reference according to the day's time, prioritizing meeting the DP. If the DP is maximum, the pressure reference is set to the control pressure (H_{ctrl}) calculated by simulation. Otherwise, the pressure reference is reduced to meet the minimum DP to minimize fluid loss without neglecting the required demand. This procedure is formally described in Algorithm 1.

Algorithm 1: Set point manager for pressure management

Input: One-day demand profile (DP), time-varying leak coefficient (λ_k), leak detection threshold (λ_{th}).

Output: Time-varying set point (s_k). Initialization:

for each index i **in** DP **do**

 Compute the control pressure (H_{ctrl}) to ensure scheduled demand in DP assuming leak-free conditions:

$$H_{ctrl}[i] \leftarrow \text{solveHydraulics}(\text{DP}[i])$$

end for

Online operation:

for each time-step k **do**

if $\lambda_k > \lambda_{th}$

$$s_k \leftarrow \min(H_{ctrl})$$

else

$$t_k \leftarrow \text{getCurrentTime}()$$

$$s_k \leftarrow \text{interp}(H_{ctrl}, t_k)$$

end if

end for

The solveHydraulics() subroutine in Algorithm 1 solves the hydraulic model to compute the required pressure in the control node, ensuring that the demand is satisfied at each hour of the day. The interp() subroutine computes by interpolation the control pressure corresponding to each time step t_k .

5. Simulation Results

This section presents simulations of the WDS built in the Hydroinformatics Laboratory at the Technological Institute of Tuxtla Gutierrez, whose mathematical model was validated in [23]. The system parameters are given in Table 1. The system can be configured as a single horizontal pipeline and a branched network as in Figures 1, 3, and 4. An EKF-based method [24] has been considered to estimate the leak position z_ℓ , its magnitude $\hat{\lambda}$, and the leak pressure H_ℓ for Cases 1 and 2. A leak occurring in an accessory (pipeline joint) located in an unknown position is assumed for Case 3; nevertheless, it can be estimated by an EKF, e.g., [25]. However, it is important to note that the proposed method is not attached to any leak estimation method, and it could be generalized to any WDS within the topology presented here.

The initial conditions for the flows $Q_i(0)$ and pressures $H_i(0)$ for all cases are presented in Table 2. The sampling period is $T_s = 0.01$ s. Gaussian noise with a variance of $2.53 \times 10^{-10} \text{ m}^6/\text{s}^2$ for the flow rate and $3.72 \times 10^{-4} \text{ m}^2$ for the pressure were added to the signals. These noise levels were characterized according to the response of the Yokogawa sensors installed in the physical system as described in [32]. Cases 1 and 2 take into account an uncertainty for z_ℓ , λ , and H_ℓ of $\pm 15\%$, that is, an error in the estimation of the leakage variables. For Case 3, the same value for λ as the previous cases is considered. For the MPC, the prediction horizon is $n_y = 15$, the control horizon is $n_u = 3$, and the weights are $\mathcal{Q} = 1$ and $\mathcal{R} = 0.5$. These values are based on well-known tuning methods [33]. The constraints were proposed according to the physical behavior and expected DP. For all cases, $0.1 \leq r \leq 1$. For Case 1, the constraints on the states are

$$1 \times 10^{-3} \text{ m}^3/\text{s} \leq Q_{1,2,3} \leq 1.3 \times 10^{-3} \text{ m}^3/\text{s},$$

$$2 \text{ m} \leq H_3 \leq 2.87 \text{ m}.$$

Table 1. System parameters.

Parameter	Value
Relative roughness (ϵ)	0.1×10^{-4}
Fluid (water) density (ρ)	995.736 kg/m^3
Kinematic viscosity (ν)	$0.803 \times 10^{-6} \text{ m}^2/\text{s}$
Leak coefficient (λ)	$0.1 \times 10^{-4} \text{ m}^{5/2}/\text{s}$
Gravity acceleration (g)	9.81 m/s^2
Valve coefficient (C_v)	1.156
Pipeline diameter (d)	0.048 m
Case 1 lengths ($z_{1,2,3}$)	11.278, 27.662, 75.7 m
Case 2 lengths ($z_{1,2,3}$)	38.94, 14.79, 60.91 m
Case 3 lengths ($z_{1,\dots,5}$)	34.456, 31.056, 34.456, 35.456, 35.456 m

Table 2. Initial conditions for the three simulation cases.

Variable	Case 1	Case 2	Case 3	Units
$H_1(0)$	3.5480	7.04	–	m
$H_2(0)$	–	5.2908	1.99	m
$H_3(0)$	3.2	–	–	m
$H_4(0)$	1.5884	2.21	–	m
$H_{B1}(0)$	–	–	5.6542	m
$H_{B2,B3,B4}(0)$	–	–	1	m
$Q_1(0)$	0.0012	0.0019	0.0030	m ³ /s
$Q_2(0)$	0.0012	0.0019	0.0014	m ³ /s
$Q_3(0)$	0.0012	0.0019	0.0007	m ³ /s
$Q_4(0)$	–	–	0.0015	m ³ /s
$Q_5(0)$	–	–	0.0007	m ³ /s
$H_\ell(0)$	3.2	4.6889	1.2255	m

For Case 2,

$$1.5 \times 10^{-3} \text{ m}^3/\text{s} \leq Q_{1,2,3} \leq 2 \times 10^{-3} \text{ m}^3/\text{s},$$

$$4.3 \text{ m} \leq H_2 \leq 5.4 \text{ m}.$$

Finally, for Case 3,

$$0.17 \times 10^{-3} \text{ m}^3/\text{s} \leq Q_1 \leq 3.2 \times 10^{-3} \text{ m}^3/\text{s},$$

$$0.7 \times 10^{-3} \text{ m}^3/\text{s} \leq Q_2 \leq 1.6 \times 10^{-3} \text{ m}^3/\text{s},$$

$$0.4 \times 10^{-3} \text{ m}^3/\text{s} \leq Q_3 \leq 0.7 \times 10^{-3} \text{ m}^3/\text{s},$$

$$0.95 \times 10^{-3} \text{ m}^3/\text{s} \leq Q_4 \leq 1.5 \times 10^{-3} \text{ m}^3/\text{s},$$

$$0.4 \times 10^{-3} \text{ m}^3/\text{s} \leq Q_5 \leq 0.7 \times 10^{-3} \text{ m}^3/\text{s},$$

$$1.2 \text{ m} \leq H_2 \leq 2 \text{ m}.$$

These constraints were chosen to satisfy a minimum demand, even in the case of a leak. The simulation covers a period of 24 h whose maximum and minimal magnitudes are concerning a typical demand profile. For Case 1, the results are displayed in Figure 7. The top plot of Figure 7 shows the pressure in the demand node. The dotted line is the DP, the dashed is the reference driven by the SPM that adapts S_k during the leak period to reduce the water losses, and the solid line represents the controlled pressure (H_3) in the node. The leak occurs at $t = 3$ h with the conditions given in Table 1. The SPM identifies a time of minimum demand and reduces the pressure. This pressure reduction remains until $t = 7$ h where the maximum demand period starts. During the period of maximum demand, the SPM does not adjust the reference because the priority is to satisfy the demand. After $t = 17$ h, the SPM again reduces S_k to reduce the leak magnitude. The middle plot of Figure 7 shows the effect of leak reduction due to the SPM. The solid line is the leak magnitude without the SPM and the dashed-line with the SPM. The bottom plot of Figure 7 shows the PRV opening to track the desired reference. It is important to note that all the constraints are satisfied for the MPC. We can conclude that the MPC tracks the set point with good performance even during the leak period, which demonstrates the effectiveness of the proposed method.

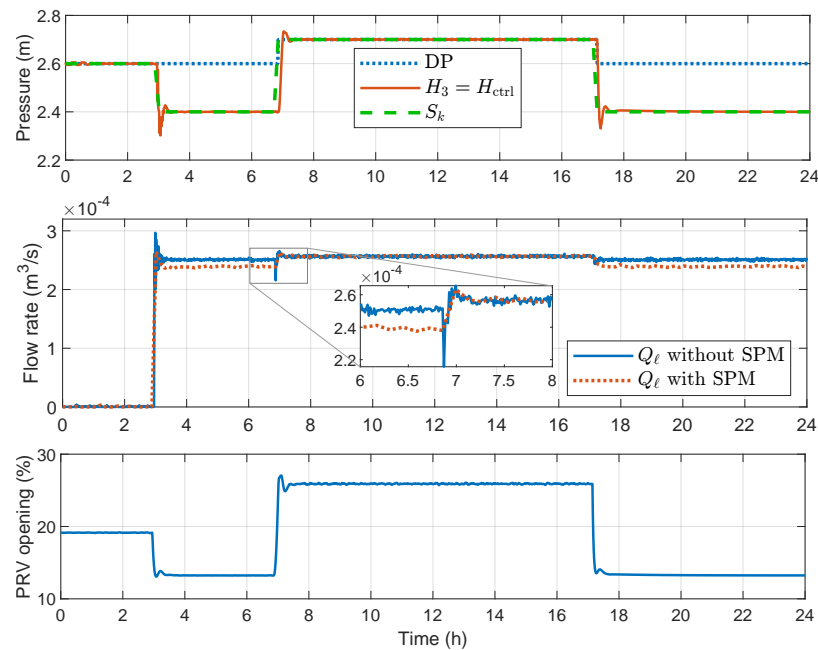


Figure 7. Case 1: Pressure control and management in a pipeline with a leak before the controlled node.

For Case 2, the results are displayed in Figure 8. As can be seen, the MPC tracks the pressure reference provided by the SPM even in a leakage scenario. Similar to Case 1, the SPM automatically adjusts the controlled node pressure at the maximum and minimum demand periods to reduce water losses, and all the constraints are satisfied for the MPC. To test the robustness of the controller, $\pm 15\%$ of uncertainty in the location of the leak is added, demonstrating the performance of the controller even when there is a leak location error.

To contrast the results with the traditional method of control and management of pressure in a pipe, Figure 9 presents a comparison between a traditional PID controller and the proposed predictive control scheme. The PID gains were computed with the Matlab[®] PIDtuner[®], obtaining the optimal values for proportional $K_P = 1$, integral $K_I = 0.5$, and derivative $K_D = 0.5$ gains. It can be seen that the PID tracks the demand profile with good performance. However, in the event of a leak, the set point remains the same, and the leak is seen by the controller as a disturbance. So the PID adjusts (increases) the pressure to track the original demand profile. This way of operating guarantees the pressure in the demand node but at the cost of fluid loss. In contrast, the MPC is aware of the leak event and with the aid of the set point manager adjusts the pressure operating point to a lower value reducing the fluid loss and delivering the minimum allowed pressure to the demand node.

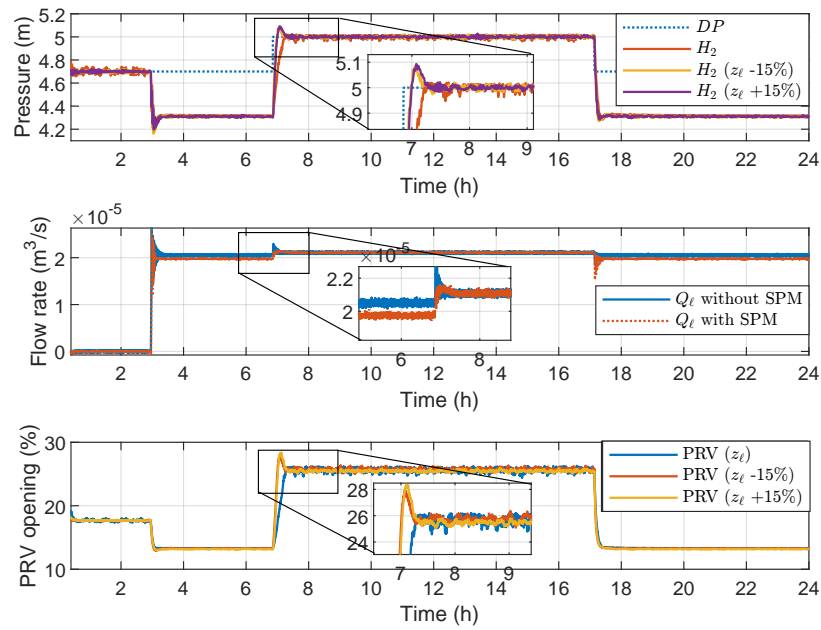


Figure 8. Case 2: Pressure control and management in a pipeline with a leak after the controlled node. Including $\pm 15\%$ of uncertainty in the leak location.

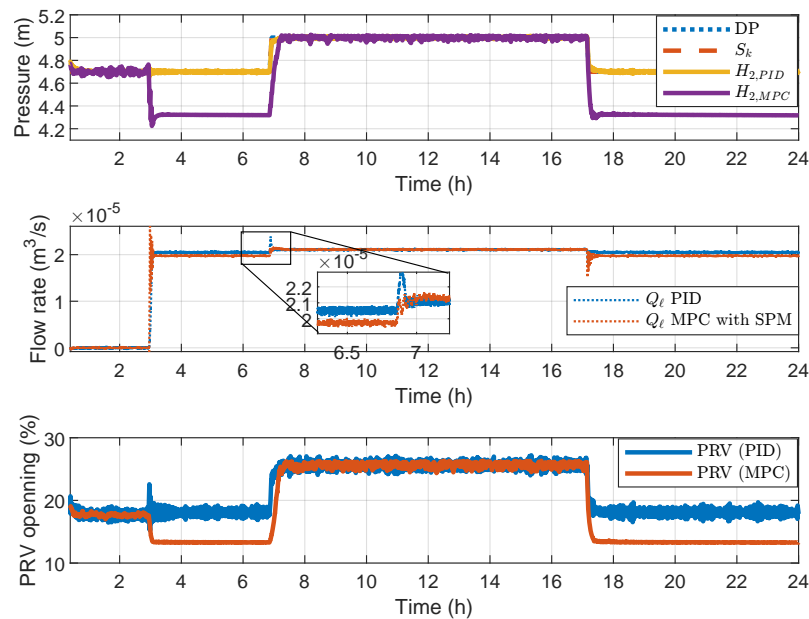


Figure 9. Case 2: Comparison between a traditional PID controller and the proposed control scheme.

Results for Case 3 are shown in Figure 10, with a leak being simulated at a connection. The MPC still achieves a good tracking of S_k that the SPM sets, which proposes a pressure reduction at the time of the leak. This pressure reduction remains constant in the first period until the maximum demand. The middle plot of Figure 10 shows the leak magnitude with and without the SPM. Note that Case 3 considers three demand nodes (due to its branches) controlled by adjusting the pressure on the controlled node. Therefore, it is essential to analyze these flows to evaluate the MPC effectiveness. The magnitudes with and without the SPM are represented by a dashed and a solid line, respectively. As can be seen, the priority is to satisfy each node demand during the maximum demand periods. However, during the minimum demand periods, the pressure is reduced, which reduces water losses.

Finally, Figure 11 displays the flow rates at the demand nodes for Case 3: Q_3 , Q_4 , and Q_5 . The solid lines are the flows without the effect of the SPM, and the dotted lines

represent the reduction due to the SPM. The reduction of flows is within the minimum permissible limits given by constraints on the states taken into account by the MPC and only during the hours of minimum demand.

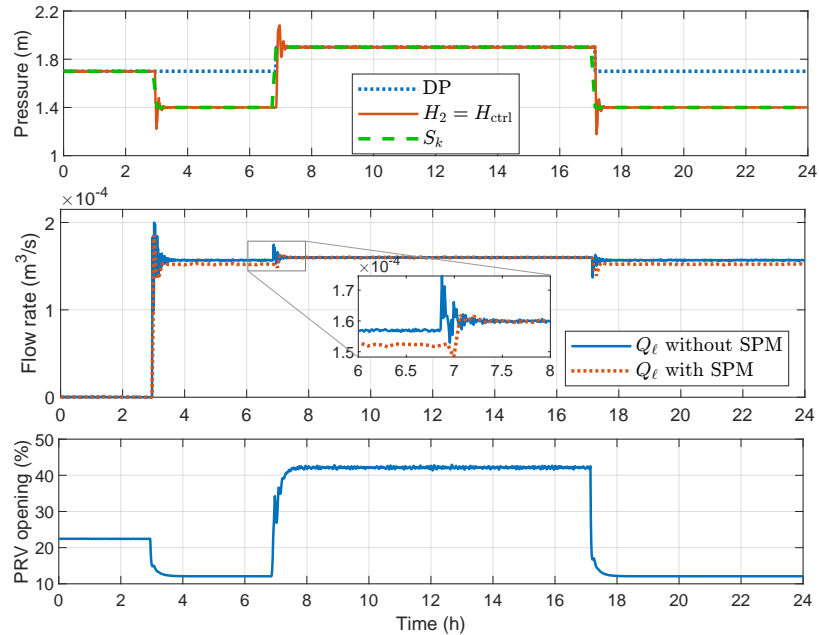


Figure 10. Case 3: Pressure control and management in a branched water distribution network.

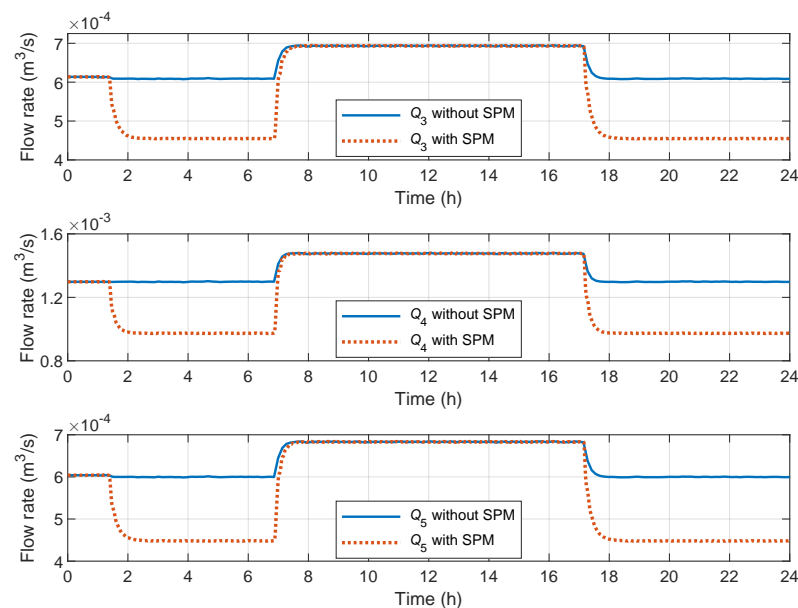


Figure 11. Case 3: Flows in the main water intakes for the WDS.

For all the cases, the trade-off between maintaining a minimum demand and reducing the leak magnitude was achieved due to the good performance of the MPC and the SPM algorithm. As a result, a reduction of water losses of $\approx 5\%$ was accomplished, which is important due to the fact that water distribution systems must operate without interruptions throughout the year.

6. Conclusions

This work proposed a model-based predictive controller for managing and controlling water distribution systems. The proposed method seeks a trade-off between maintaining

water supply and reducing water losses due to leaks. To achieve this goal, the MPC is based on the water-hammer equations of the hydraulic system together with physical constraints and an adaptive demand profile managed by a set point manager algorithm. The control input was calculated by minimizing the output errors with respect to the demand profile driven by the set point manager. The control objective was formulated with state, input, and output constraints on the cost function. Moreover, the control scheme includes a strategy to handle the strong nonlinear behavior of the PRV. The controller was tested with numerical simulations in a model characterized by a real pipeline and pipe network located at the Hydroinformatics Laboratory of the Technological Institute of Tuxtla Gutiérrez. Therefore, this work presents a mathematical model based on a real system and realistic operating conditions. The simulation results illustrate the performance and robustness of the MPC for pressure management in the system and the reduction of leaks due to water losses, with an average of 5% in the presence of noise, disturbances, and uncertainty in the leak location estimate. Future work will extend this work to an integrated methodology of a multi-leak tolerant control algorithm.

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